

Offshore Hydromechanics Part 2

Ir. Peter Naaijen

1. Introduction and Ship Motion

OE 4630 2012 - 2013
Offshore Hydromechanics, lecture 1



[1]



[2]

Take your laptop, i- or whatever smart-phone and go to:
www.rwpoll.com
Login with session ID

Teacher module II:

- Ir. Peter Naaijen
- p.naijen@tudelft.nl
- Room 34 B-0-360 (next to towing tank)

Book:

- Offshore Hydromechanics, by J.M.J. Journee & W.W.Massie

Useful weblinks:

- <http://www.shipmotions.nl>
- Blackboard

OE4630 module II course content

- +/- 7 Lectures
- Bonus assignments (optional, contributes 20% of your exam grade)
- Laboratory Exercise (starting 30 nov)
 - 1 of the bonus assignments is dedicated to this exercise
 - Groups of 7 students
 - Subscription available soon on BB
- Written exam

Schedule OE4630 D2, Offshore Hydromechanics Pt 2, 2012-2013 **Version 1 (9-11-2012)**
Disclaimer: always track for (last minute) changes in location at huisgeroosters.tudelft.nl/

Date:	Time:	Type:	Teacher:	Location
Wed 14 Nov	13.30 – 16.30	Lecture	Peter Naaijen	3mE-C2 D (James Watt)
Wed 14 Nov	16.30 – 17.30	Assignment assistance /Questions	Peter Naaijen	3mE-C2 D (James Watt)
Fri 16 Nov	10.30 – 12.30	Lecture	Peter Naaijen	3mE-C2 B (Isaac Newton)
Mon 19 Nov	15.30 – 17.30	Lecture	Peter Naaijen	3mE-C2 B (Isaac Newton)
Tue 20 Nov	13.30 – 15.30	Assignment assistance /Questions	Peter Naaijen	3mE-C2 C (Daniel Bernoulli)
Wed 28 Nov	13.30 – 15.30	Lecture	Peter Naaijen	3mE-C2 D (James Watt)
Wed 28 Nov	15.30 – 17.30	Assignment assistance /Questions	Peter Naaijen	3mE-C2 D (James Watt)
Fri 30 Nov	10.30 – 13.00	Lab session	Peter Naaijen	Towing Tank
Mon 3 Dec	15.30 – 17.30	Lecture	Peter Naaijen	3mE-C2 B (Isaac Newton)
Tue 4 Dec	13.30 – 16.00	Lab session	Gideon Hertzberger	Towing Tank
Tue 4 Dec	16.30 – 17.30	Assignment assistance /Questions	Peter Naaijen	Room Peter Naaijen (34 B 0 360)
Mon 10 Dec	15.30 – 17.30	Lecture	Peter Naaijen	3mE-C2 B (Isaac Newton)
Mon 17 Dec	15.30 – 17.30	Lecture	Peter Naaijen	3mE-C2 B (Isaac Newton)
Mon 7 Jan	15.30 – 17.30	Lecture	Peter Naaijen	3mE-C2 B (Isaac Newton)

Lecture notes:

- Disclaimer: Not everything you (should) learn is in the lecture notes (lees: niet alles wat op het tentamen gevraagd kan worden staat in diktaat...) →

Make personal notes during lectures!!

- Don't save your questions 'till the break →

Ask if anything is unclear

Learning goals Module II, behavior of floating bodies in waves

• Definition of ship motions

Motion Response in regular waves:

- How to use RAO's
- Understand the terms in the equation of motion: hydromechanic reaction forces, wave exciting forces
- How to solve RAO's from the equation of motion

Motion Response in irregular waves:

- How to determine response in irregular waves from RAO's and wave spectrum without forward speed

3D linear Potential Theory

- How to determine hydrodynamic reaction coefficients and wave forces from Velocity Potential
- How to determine Velocity Potential

Motion Response in irregular waves:

- How to determine response in irregular waves from RAO's and wave spectrum with forward speed

Ch. 8

- Make down time analysis using wave spectra, scatter diagram and RAO's

Structural aspects:

- Calculate internal forces and bending moments due to waves

Nonlinear behavior:

- Calculate mean horizontal wave force on wall
- Use of time domain motion equation

Ch.6

Introduction



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Introduction

Offshore → oil resources have to be explored in deeper water → floating structures instead of bottom founded



[4]

Introduction

Reasons to study waves and ship behavior in waves:

- the dynamic loads on the floating structure, its cargo or its equipment:
 - Inertia forces on sea fastening due to accelerations:



Introduction

Reasons to study waves and ship behavior in waves:

- the dynamic loads on the floating structure, its cargo or its equipment:
 - Direct wave induced structural loads

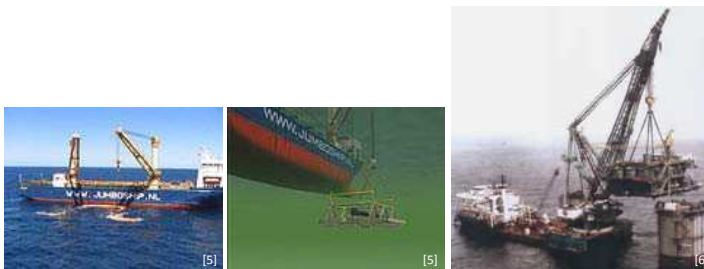


Minimum required air gap to avoid wave damage

Introduction

Reasons to study waves and ship behavior in waves:

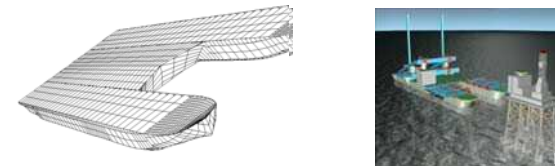
- Determine allowable / survival conditions for offshore operations



Introduction

Decommissioning / Installation / Pipe laying → Excalibur / Allseas 'Pieter Schelte'

- Motion Analysis



Introduction

Reasons to study waves and ship behavior in waves:

- the dynamic loads on the floating structure, its cargo or its equipment:
 - Forces on mooring system, motion envelopes loading arms



[8]



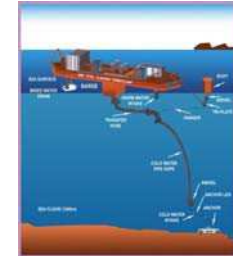
[2]

Introduction

Floating Offshore: More than just oil



Floating wind farm [9]



OTEC [10]

Introduction

Floating Offshore: More than just oil



[11]



[12]

Wave energy conversion

Introduction

Floating Offshore: More than just oil



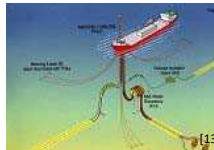
Mega Floaters

Introduction

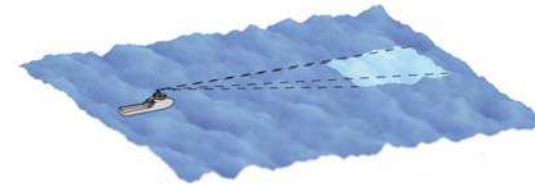
Reasons to study waves and ship behavior in waves:

- Determine allowable / survival conditions for offshore operations
- Downtime analysis

Wave Direction (°)	Wave Period (s)															
	35	45	55	65	75	85	95	105	115	125	135	145	155	165	175	185
14	0	0	0	0	2	3	5	8	12	18	28	42	60	85	120	165
13	0	0	0	0	3	5	8	12	18	28	42	60	85	120	165	210
12	0	0	0	0	7	12	18	28	42	60	85	120	165	210	255	300
11	0	0	0	0	17	30	45	65	90	120	165	210	255	300	345	390
10	0	0	0	1	41	75	110	150	195	240	285	330	375	420	465	510
9	0	0	0	4	139	265	400	540	685	835	985	1135	1285	1435	1585	1735
8	0	0	0	12	285	565	850	1135	1420	1705	1990	2275	2560	2845	3130	3415
7	0	0	0	41	985	1970	2955	3940	4925	5910	6895	7880	8865	9850	10835	11820
6	0	0	1	138	2760	5520	8280	11040	13800	16560	19320	22080	24840	27600	30360	33120
5	0	0	7	471	942	1884	2826	3768	4710	5652	6594	7536	8478	9420	10362	11304
4	0	0	38	1986	3972	7944	11916	15888	19860	23832	27804	31776	35748	39720	43692	47664
3	0	0	148	5076	10152	15228	20304	25380	30456	35532	40608	45684	50760	55836	60912	65988
2	0	4	488	15948	31896	47844	63792	79740	95688	111636	127584	143532	159480	175428	191376	207324
1	0	40	2838	8814	17628	26442	35256	44070	52884	61698	70512	79326	88140	96954	105768	114582
0	5	30	384	576	864	1152	1440	1728	2016	2304	2592	2880	3168	3456	3744	4032
Total	5	30	688	1032	1376	1720	2064	2408	2752	3096	3440	3784	4128	4472	4816	5160



Real-time motion prediction
Using X-band radar remote wave observation



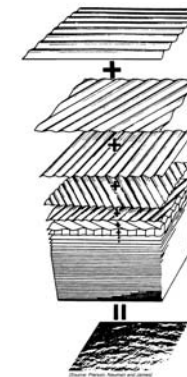
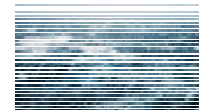
Definitions & Conventions

Regular waves
Ship motions



Irregular wind waves

apparently irregular but can be considered as a superposition of a finite number of regular waves, each having own frequency, amplitude and propagation direction



Regular waves

(Ch.5 revisited)

Regular waves

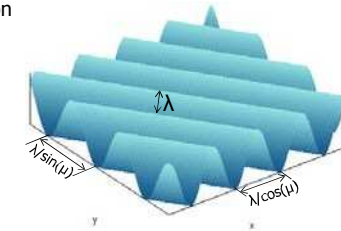
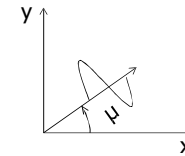
regular wave propagating in direction μ :

$$\zeta(t, x) = \zeta_a \cos(\omega t - kx \cos \mu - ky \sin \mu)$$

$$k = 2\pi / \lambda$$

$$\omega = 2\pi / T$$

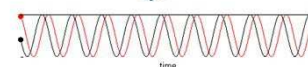
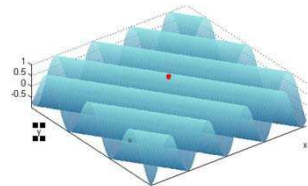
Linear solution Laplace equation



- Regular waves
- regular wave propagating in direction μ

$$\zeta(t, x) = \zeta_a \cos(\omega t - kx \cos \mu - ky \sin \mu)$$

Phase angle ϵ



Phase angle wave at black dot with respect to wave at red dot:
 $\epsilon_{\zeta, z} = -k(x-x_0) \cos \mu - k(y-y_0) \sin \mu$

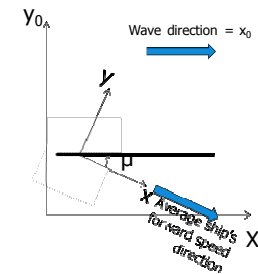
Co-ordinate systems

Definition of systems of axes

Earth fixed: (x_0, y_0, z_0)

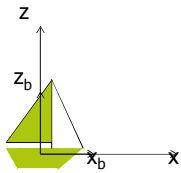
Following average ship position: (x, y, z)

wave direction with respect to ship's axes system: μ



Behavior of structures in waves

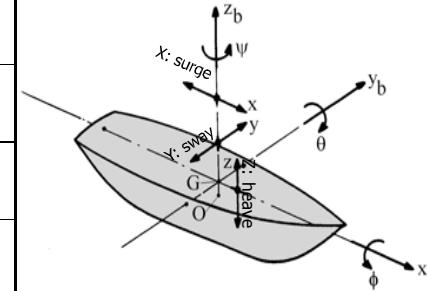
Ship's body bound axes system (x_b, y_b, z_b) follows all ship motions



Behavior of structures in waves

Definition of translations

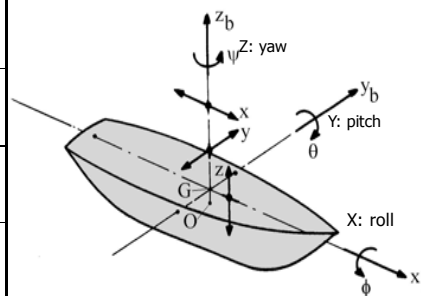
		NE	EN
1	x	Schrikken	Surge
2	y	Verzetten	Sway
3	z	Dampen	Heave



Behavior of structures in waves

Definition of rotations

		NE	EN
4	x	Slingeren	Roll
5	y	Stampen	Pitch
6	z	Gieren	Yaw



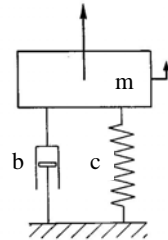
How do we describe ship motion response?

Rao's
Phase angles

Mass-Spring system:

$$m\ddot{z} + b\dot{z} + cz = F_a \cos(\omega t) \quad \text{Motion equation}$$

$$z(t) = z_a \cos(\omega t + \varepsilon) \quad \text{Steady state solution}$$



Motions of and about COG

Surge (*schrikken*): $x = x_a \cos(\omega t + \varepsilon_{x\zeta})$ Amplitude Phase angle

Sway (*verzetten*): $y = y_a \cos(\omega t + \varepsilon_{y\zeta})$

Heave (*dompen*): $z = z_a \cos(\omega t + \varepsilon_{z\zeta})$

Roll (*rollen*): $\langle \text{phi} \rangle \phi = \phi_a \cos(\omega t + \varepsilon_{\phi\zeta})$

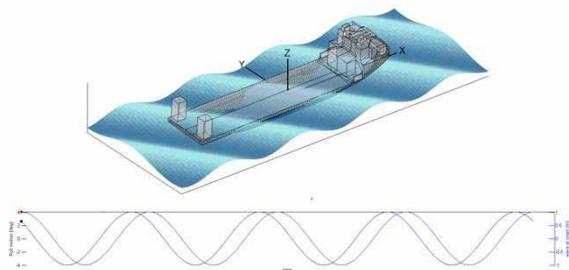
Pitch (*stampen*): $\langle \text{theta} \rangle \theta = \theta_a \cos(\omega t + \varepsilon_{\theta\zeta})$

Yaw (*gieren*): $\langle \text{psi} \rangle \psi = \psi_a \cos(\omega t + \varepsilon_{\psi\zeta})$

Phase angles ε are related to undisturbed wave at origin of steadily translating ship-bound system of axes (\rightarrow COG)

Motions of and about COG

Phase angles ε are related to undisturbed wave at origin of steadily translating ship-bound system of axes (\rightarrow COG)



Motions of and about COG

Surge (*schrikken*): $x = x_a \cos(\omega t + \varepsilon_{x\zeta})$ $RAOSurge = \frac{x}{\zeta_a} \omega \mu$

Sway (*verzetten*): $y = y_a \cos(\omega t + \varepsilon_{y\zeta})$ $RAOSway = \frac{y_a}{\zeta_a}(\omega, \mu)$

Heave (*dompen*): $z = z_a \cos(\omega t + \varepsilon_{z\zeta})$ $RAOHeave = \frac{z_a}{\zeta_a}(\omega, \mu)$

Roll (*rollen*): $\langle \text{phi} \rangle \phi = \phi_a \cos(\omega t + \varepsilon_{\phi\zeta})$ $RAORoll = \frac{\phi_a}{\zeta_a}(\omega, \mu)$

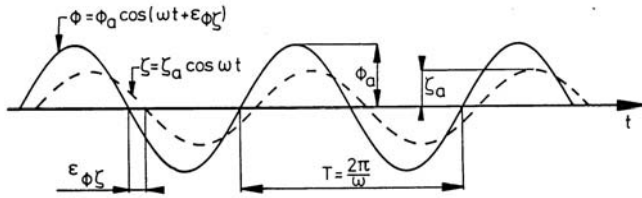
Pitch (*stampen*): $\langle \text{theta} \rangle \theta = \theta_a \cos(\omega t + \varepsilon_{\theta\zeta})$ $RAOPitch = \frac{\theta_a}{\zeta_a}(\omega, \mu)$

Yaw (*gieren*): $\langle \text{psi} \rangle \psi = \psi_a \cos(\omega t + \varepsilon_{\psi\zeta})$ $RAOYaw = \frac{\psi_a}{\zeta_a}(\omega, \mu)$

RAO and phase depend on:

- Wave frequency
- Wave direction

Example: roll signal



Displacement $\phi = \phi_a \cos(\omega_e t + \epsilon_{\phi\zeta})$
 Velocity... $\dot{\phi} = -\omega \phi_a \sin(\omega_e t + \epsilon_{\phi\zeta}) = \omega \phi_a \cos(\omega_e t + \epsilon_{\phi\zeta} + \pi/2)$
 Acceleration... $\ddot{\phi} = -\omega^2 \phi_a \cos(\omega_e t + \epsilon_{\phi\zeta}) = \omega_e^2 \phi_a \cos(\omega_e t + \epsilon_{\phi\zeta} + \pi)$

Motions of and about COG

- 1 Surge(schrikken): $x = x_a \cos(\omega_e t + \epsilon_{x\zeta})$
- 2 Sway(verzetten): $y = y_a \cos(\omega_e t + \epsilon_{y\zeta})$
- 3 Heave(dampen): $z = z_a \cos(\omega_e t + \epsilon_{z\zeta})$
- 4 Roll(rollen): $(\text{phi}) \phi = \phi_a \cos(\omega_e t + \epsilon_{\phi\zeta})$
- 5 Pitch(stampen): $(\text{theta}) \theta = \theta_a \cos(\omega_e t + \epsilon_{\theta\zeta})$
- 6 Yaw(gieren): $(\text{psi}) \psi = \psi_a \cos(\omega_e t + \epsilon_{\psi\zeta})$

- Frequency of input (regular wave) and output (motion) is ALWAYS THE SAME !!
- Phase can be positive ! (Shipmotion ahead of wave elevation at COG)
- Due to symmetry: some of the motions will be zero
- Ratio of motion amplitude / wave amplitude = **RAO** (Response Amplitude Operator)
- RAO's and phase angles depend on wave frequency and wave direction
- RAO's and phase angles must be calculated by dedicated software or measured by experiments
- Only some special cases in which 'common sense' is enough:

Consider Long waves relative to ship dimensions

What is the RAO of pitch in head waves ?

- Phase angle heave in head waves ?...
- RAO pitch in head waves ?...
- Phase angle pitch in head waves ?...
- Phase angle pitch in following waves ?...

Local motions (in steadily translating axes system)

- **Only variations!!**
- **Linearized!!**

$$\begin{pmatrix} x_p(t) \\ y_p(t) \\ z_p(t) \end{pmatrix} = \begin{pmatrix} x(t) \\ y(t) \\ z(t) \end{pmatrix} + \begin{pmatrix} 0 & -\psi(t) & \theta(t) \\ \psi(t) & 0 & -\phi(t) \\ -\theta(t) & \phi(t) & 0 \end{pmatrix} \begin{pmatrix} x_{bP} \\ y_{bP} \\ z_{bP} \end{pmatrix}$$

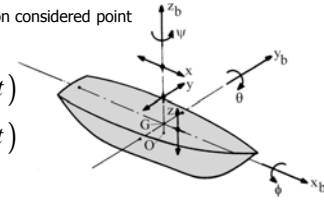
6 DOF Ship motions

Location considered point

$$x_p(t) = x(t) - y_{bP}\psi(t) + z_{bP}\theta(t)$$

$$y_p(t) = y(t) + x_{bP}\psi(t) - z_{bP}\phi(t)$$

$$z_p(t) = z(t) - x_{bP}\theta(t) + y_{bP}\phi(t)$$



Local Motions

Example 3: horizontal crane tip motions

The tip of an onboard crane, location:
 $x_b, y_b, z_b = -40, -9.8, 25.0$



For a frequency $\omega=0.6$ the RAO's and phase angles of the ship motions are:

SURGE		SWAY		HEAVE		ROLL		PITCH		YAW	
RAO	phase	RAO	phase	RAO	phase	RAO	phase	RAO	phase	RAO	phase
-	degr	-	degr	-	degr	deg/m	degr	deg/m	degr	deg/m	degr
1.014E-03	3.421E+02	5.992E-01	2.811E+02	9.991E-01	3.580E+02	2.590E+00	1.002E+02	2.424E-03	1.922E+02	2.102E-04	5.686E+01

Calculate the RAO and phase angle of the transverse horizontal motion (y-direction) of the crane tip.

Complex notation of harmonic functions

$$1 \text{ Surge (schrikken): } x = x_a \cos(\omega t + \varepsilon_{x_z})$$

$$= \text{Re} \left(x_a e^{i(\omega t + \varepsilon_{x_z})} \right)$$

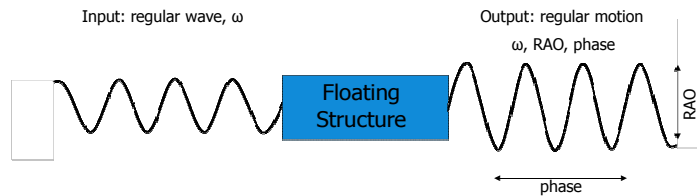
$$= \text{Re} \left(x_a e^{\varepsilon_{x_z}} e^{i\omega t} \right)$$

Complex motion amplitude

$$= \text{Re} \left(\hat{x}_a e^{i\omega t} \right)$$

Relation between Motions and Waves

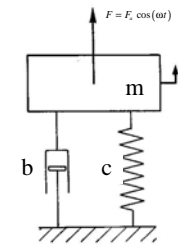
How to calculate RAO's and phases ?



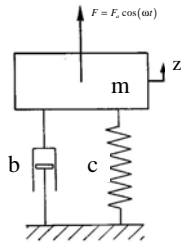
Mass-Spring system:

Forces acting on body:

...?



Mass-Spring system:



$$m\ddot{z} + b\dot{z} + cz = F_a \cos(\omega t)$$

Transient solution

$$z(t) = A_1 e^{-\zeta \omega_0 t} \sin(\sqrt{1-\zeta^2} \omega_0 t + \phi_1)$$

$$\left(\zeta = \frac{b}{2\sqrt{mc}}\right) \text{ Damping ratio}$$

Steady state solution:

$$z(t) = z_a \cos(\omega t + \varepsilon)$$

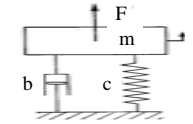
$$\varepsilon = a \tan\left(\frac{-b\omega}{(-\omega^2(m+c))}\right)$$

$$z_a = \frac{F_a}{\sqrt{((-m)\omega^2 + c)^2 + (b\omega)^2}}$$

Moving ship in waves:



[14]



$$m\ddot{z} + b\dot{z} + c_3 z = F_{a3} \cos(\omega t)$$

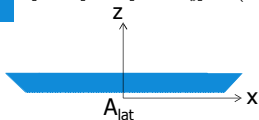
Restoring coefficient for heave ?

$$m\ddot{\phi} + b_4 \dot{\phi} + c_4 \phi = F_{a4} \cos(\omega t)$$

Restoring coefficient for roll ?
m for roll ?

What is the hydrostatic spring coefficient for the sway motion ?

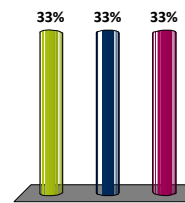
$$m_2 \ddot{y} + b_2 \dot{y} + c_2 y = F_{a2} \cos(\omega t)$$



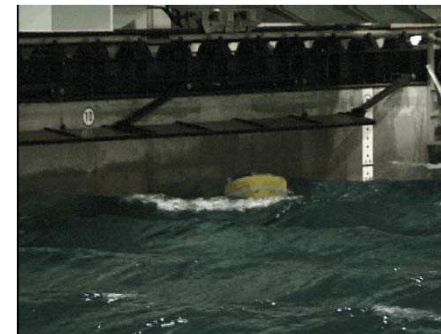
A. $c_2 = A_{wl} \rho g$

B. $c_2 = A_{lat} \rho g$

C. $c_2 = 0$



Non linear stability issue...



Floating stab.

Stability moment

$$M_x = \rho g \nabla \cdot GZ_{\phi} = \rho g \nabla \cdot GM \sin \phi = \rho g \nabla \cdot GM \cdot \phi$$

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Marine Engineering, Ship Hydromechanics Section

Moving ship in waves:

$$m_4 \ddot{\phi} + b_4 \dot{\phi} + c_4 \phi = F_{a4} \cos(\omega t)$$

Restoring coefficient for roll ?

Rotation around COF

Rotation around COG
= Rotation around COF
+ vertical translation $dz = FG - FG \cos \phi \approx 0$
+ horizontal translation $dy = FG \sin \phi \approx FG \phi$

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Moving ship in waves: Not in air but in water!

$F = m \cdot z$

SHIP MOTION : HEAVE

- F_w
- $-c \cdot z$
- $-b \cdot \dot{z}$
- $-a \cdot z$ (Only potential / wave damping)

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Moving ship in waves:

$$(m + a) \cdot \ddot{z} + b \cdot \dot{z} + c \cdot z = F_w$$

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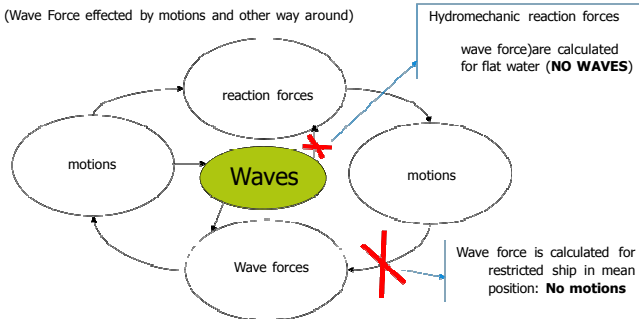
Moving ship in waves:



Solutions

$$(m+a)\ddot{z} + b\dot{z} + c \cdot z = F_w$$

2. (Wave Force effected by motions and other way around)



Back to Regular waves

regular wave propagating in direction μ
 $\zeta(t, x) = \zeta_a \cos(\omega t - kx \cos \mu - ky \sin \mu)$

Linear solution Laplace equation

In order to calculate forces on immersed bodies:
 What happens underneath free surface ?

Back to Regular waves

regular wave propagating in direction μ
 $\zeta(t, x) = \zeta_a \cos(\omega t - kx \cos \mu - ky \sin \mu)$

Linear solution Laplace equation

In order to calculate forces on immersed bodies:
 What happens underneath free surface ?

Potential Theory

What is potential theory ?:
 way to give a mathematical description of flowfield

Most complete mathematical description of flow is
 viscous Navier-Stokes equation:

Navier-Stokes vergelijkingen:

$$\rho \frac{\partial u}{\partial t} + \rho u \frac{\partial u}{\partial x} + \rho v \frac{\partial u}{\partial y} + \rho w \frac{\partial u}{\partial z} = -\frac{\partial p}{\partial x} + \frac{\partial}{\partial x} \left(\lambda \nabla \cdot V + 2\mu \frac{\partial u}{\partial x} \right) + \frac{\partial}{\partial y} \left[\mu \left(\frac{\partial v}{\partial x} + \frac{\partial u}{\partial y} \right) \right] + \frac{\partial}{\partial z} \left[\mu \left(\frac{\partial u}{\partial z} + \frac{\partial w}{\partial x} \right) \right]$$

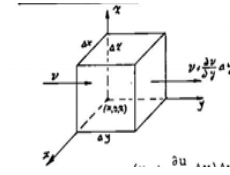
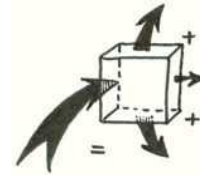
$$\rho \frac{\partial v}{\partial t} + \rho u \frac{\partial v}{\partial x} + \rho v \frac{\partial v}{\partial y} + \rho w \frac{\partial v}{\partial z} = -\frac{\partial p}{\partial y} + \frac{\partial}{\partial x} \left[\mu \left(\frac{\partial v}{\partial x} + \frac{\partial u}{\partial y} \right) \right] + \frac{\partial}{\partial y} \left(\lambda \nabla \cdot V + 2\mu \frac{\partial v}{\partial y} \right) + \frac{\partial}{\partial z} \left[\mu \left(\frac{\partial v}{\partial z} + \frac{\partial w}{\partial y} \right) \right]$$

$$\rho \frac{\partial w}{\partial t} + \rho u \frac{\partial w}{\partial x} + \rho v \frac{\partial w}{\partial y} + \rho w \frac{\partial w}{\partial z} = -\frac{\partial p}{\partial z} + \frac{\partial}{\partial x} \left[\mu \left(\frac{\partial u}{\partial z} + \frac{\partial w}{\partial x} \right) \right] + \frac{\partial}{\partial y} \left[\mu \left(\frac{\partial v}{\partial z} + \frac{\partial w}{\partial y} \right) \right] + \frac{\partial}{\partial z} \left(\lambda \nabla \cdot V + 2\mu \frac{\partial w}{\partial z} \right)$$

(A-relaxed !)

→

Apply principle of continuity on control volume:



Continuity: what comes in,
must go out

This results in continuity equation:

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} + \frac{\partial w}{\partial z} = 0$$

If in addition the flow is considered to be irrotational and non
viscous →

Velocity potential function can be used to describe
water motions

Main property of velocity potential function:

for potential flow, a function $\Phi(x,y,z,t)$ exists whose derivative in a
certain arbitrary direction equals the flow velocity in that
direction. This function is called the velocity potential.

From definition of velocity potential:

$$u = \frac{\partial \Phi}{\partial x}, v = \frac{\partial \Phi}{\partial y}, w = \frac{\partial \Phi}{\partial z}$$

Substituting in continuity equation:

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} + \frac{\partial w}{\partial z} = 0$$

Results in Laplace equation:

$$\frac{\partial^2 \Phi}{\partial x^2} + \frac{\partial^2 \Phi}{\partial y^2} + \frac{\partial^2 \Phi}{\partial z^2} = 0$$

Summary

- Potential theory is mathematical way to describe flow

Important facts about velocity potential function Φ :

- definition: Φ is a function whose derivative in any direction equals the flow velocity in that direction
- Φ describes non-viscous flow
- Φ is a scalar function of space and time (NOT a vector!)

Summary

- Velocity potential for regular wave is obtained by
 - Solving Laplace equation satisfying:
 1. Seabed boundary condition
 2. Dynamic free surface condition

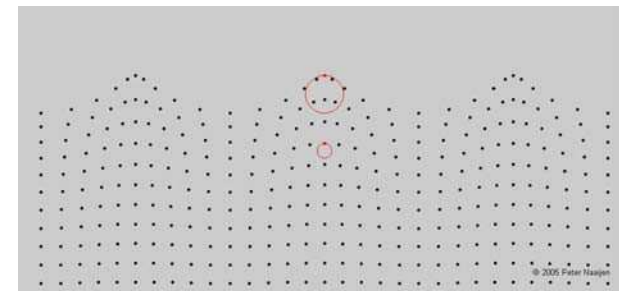
$$\Phi(x, y, z, t) = \frac{\zeta_a g}{\omega} \cdot e^{kz} \cdot \sin(kx \cos \mu + ky \sin \mu - \omega t)$$

$$\Phi(x, y, z, t) = \frac{\zeta_a g}{\omega} \cdot \frac{\cosh(k(h+z))}{\cosh(kh)} \cdot \sin(kx \cos \mu + ky \sin \mu - \omega t)$$

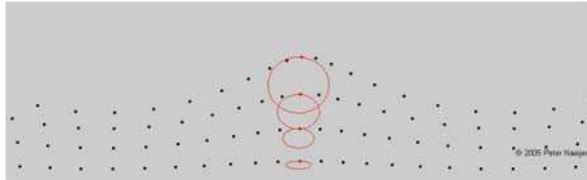
3. Kinematic free surface boundary condition results in:
Dispersion relation = relation between wave frequency and wave length

$$\omega^2 = kg \tanh(kh)$$

Water Particle Kinematics trajectories of water particles in infinite water depth



Water Particle Kinematics trajectories of water particles in finite water depth



Pressure

Pressure in the fluid can be found using Bernoulli equation
for unsteady flow:

$$\frac{\partial \Phi}{\partial t} + \frac{1}{2}(u^2 + w^2) + \frac{p}{\rho} + gz = 0$$

$$p = -\rho \frac{\partial \Phi}{\partial t} - \frac{1}{2} \rho (u^2 + w^2) - \rho gz$$

1st order fluctuating
pressure

2nd order (small
quantity
squared=small enough
to neglect)

Hydrostatic pressure
(Archimedes)

Potential Theory

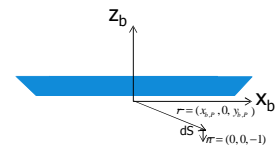
From all these velocity potentials we can derive:

- Pressure
- Forces and moments can be derived from pressures:

$$\vec{F} = -\iint_S (p \cdot \vec{n}) dS$$

$$\vec{M} = -\iint_S p \cdot (\vec{r} \times \vec{n}) dS$$

Verify these formulae (incl the signs!) yourself in order to understand them. Just check e.g. the force in heave direction (F_z) and the pitch moment (M_x) induced by a pressure on an infinite piece of hull surface dS at location P :

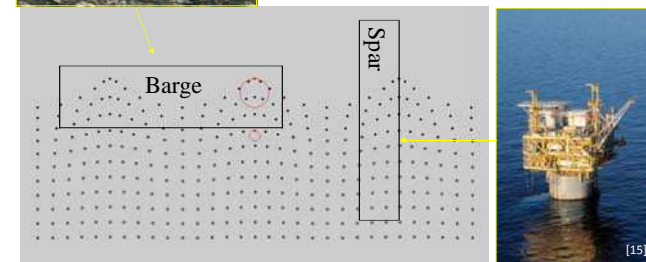


Wave Force



Determination F_w

- Froude Krilov
- Diffraction

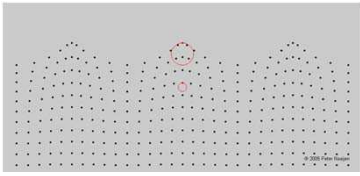


Flow superposition

Considering a fixed structure (ignoring the motions) we will try to find a description of the disturbance of the flow by the presence of the structure in the form of a velocity potential. We will call this one the diffraction potential and added to the undisturbed wave potential (for which we have an analytical expression) it will describe the total flow due to the waves.


$$(m+a)z+b+c \cdot z = F_w$$

1. Flow due to Undisturbed wave

$$\Phi_0 = -\frac{\zeta_a g}{\omega} \cdot e^{kz} \cdot \sin(\omega t - kx \cos \mu - ky \sin \mu)$$

2. Flow due to Diffraction

$$\Phi_7$$

Has to be solved. What is boundary condition at body surface ?

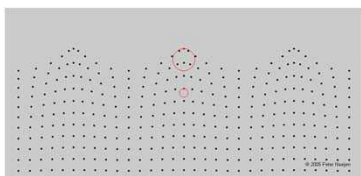


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Exciting force due to waves


$$(m+a)z+b+c \cdot z = F_w = (F_{FK}) + (F_D)$$

1. Undisturbed wave force (Froude-Krilov)

$$\Phi_0 = -\frac{\zeta_a g}{\omega} \cdot e^{kz} \cdot \sin(\omega t - kx \cos \mu - ky \sin \mu + \varepsilon)$$

2. Diffraction force

$$\Phi_7$$

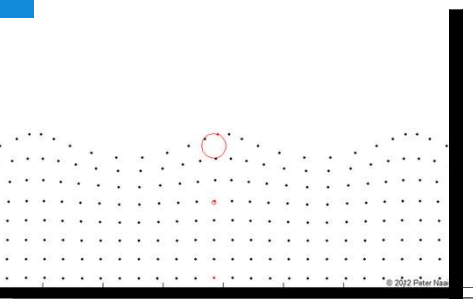
Has to be solved. What is boundary condition at body surface ?



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Wave Forces

Wave force acting on vertical wall



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Force on the wall

$$\vec{F} = - \int_{-\infty}^0 p \cdot \vec{n} dz$$

$$\Phi_0 = \frac{\zeta_a g}{\omega} e^{kz} \sin(kx - \omega t), \Phi_7 = -\frac{\zeta_a g}{\omega} e^{kz} \sin(kx + \omega t)$$

$$p = -\rho \frac{\partial \Phi}{\partial t} = -\rho \frac{\partial (\Phi_0 + \Phi_7)}{\partial t} =$$

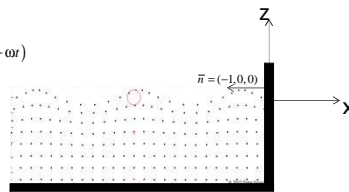
$$-\rho \frac{\partial \left(-2 \frac{\zeta_a g}{\omega} \cdot e^{kz} \sin(\omega t) \cos(kx) \right)}{\partial t} =$$

$$2\rho \zeta_a g \cdot e^{kz} \cos(kx) \cos(\omega t)$$

$$\vec{n} = (-1, 0, 0)$$

$$x = 0$$

$$F_x = \int_{-\infty}^0 2\rho \zeta_a g \cdot e^{kz} \cos(\omega t) dz = \left[2\rho \frac{\zeta_a g}{k} \cdot e^{kz} \cos(\omega t) \right]_{-\infty}^0 =$$

$$2\rho \frac{\zeta_a g}{k} \cdot \cos(\omega t) - 0$$


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Sources images

- [1] Towage of SSSR Transocean Amirante, source: Transocean
- [2] Tower Mooring, source: unknown
- [3] Rogue waves, source: unknown
- [4] Bluewater Rig No. 1, source: Friede & Goldman, LTD/GNU General Public License
- [5] Source: unknown
- [6] Rig Neptune, source: Seafarer Media
- [7] Pieter Schelte vessel, source: Excalibur
- [8] FPSO design basis, source: Statoil
- [9] Floating wind turbines, source: Principle Power Inc.
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- [12] A Pelamis installed at the Agucadoura Wave Park off Portugal, source: S.Portland/Wikipedia
- [13] Schematic of Curlew Field, United Kingdom, source: offshore-technology.com
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