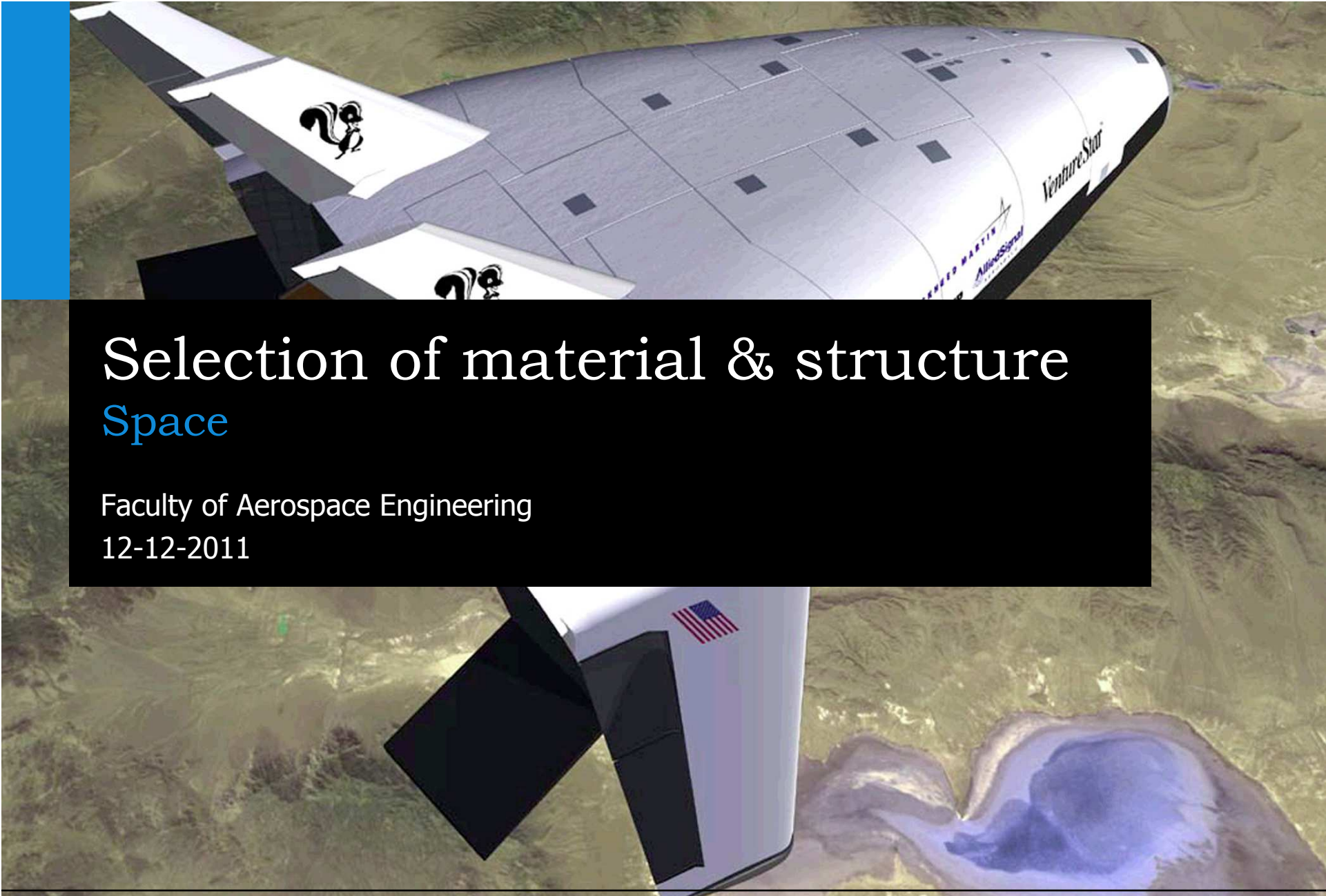


Introduction to Aerospace Engineering

Lecture slides



Selection of material & structure Space

Faculty of Aerospace Engineering
12-12-2011



Learning objectives

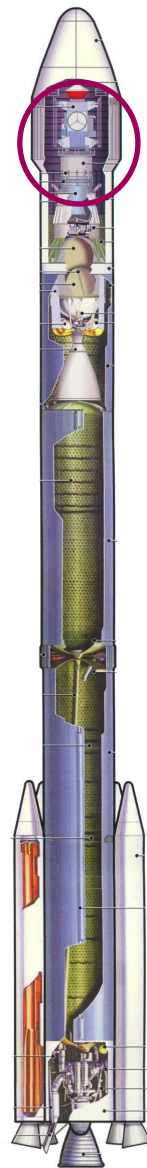
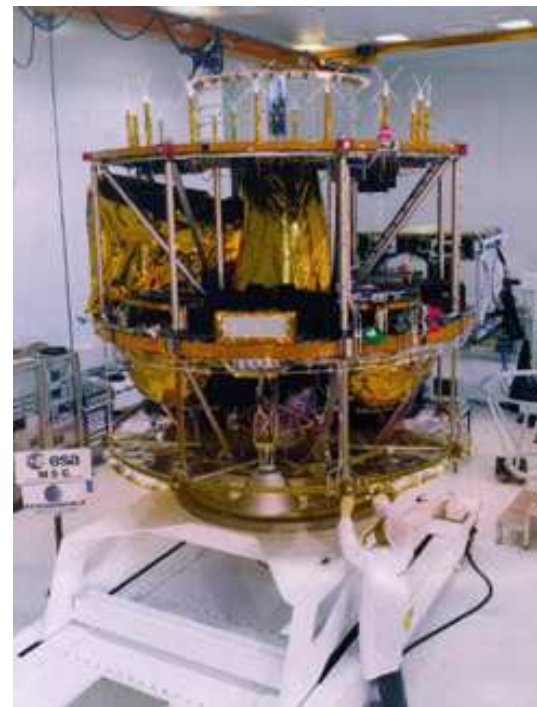
Student should be able to...

- Describe the basic steps in dimensioning a spacecraft structure
- Design simplified structure for minimum natural frequency requirements

Spacecraft structures

Typical structures

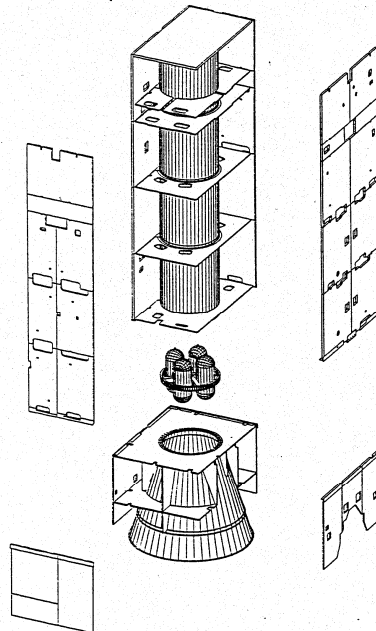
- Spacecraft
 - Struts
 - Central cylinder
- Launch vehicle
 - Fairings
 - Stage structure
 - Thrust structure
 - Adaptors



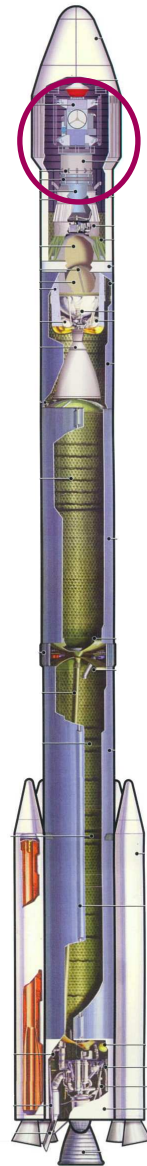
Spacecraft structures

Typical structures

- Spacecraft
 - Struts
 - Central cylinder
- Launch vehicle
 - Fairings
 - Stage structure
 - Thrust structure
 - Adaptors
- Characteristics
 - Central thrust-load-bearing member (cone/cylinder)
 - All systems attached at strong points directly, or by combinations of Struts/Platforms/Shear webs



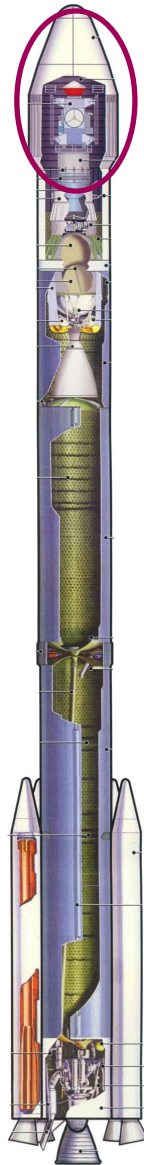
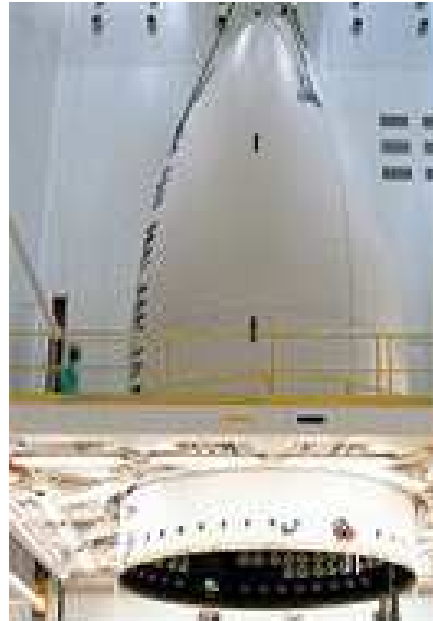
See Envisat in study collection!



Spacecraft structures

Typical structures

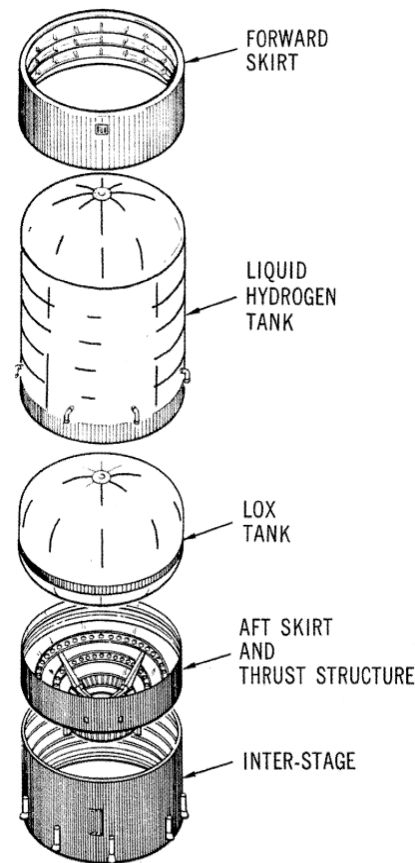
- Spacecraft
- Launch vehicle
 - Fairings
 - Stage structure
 - Thrust structure
 - Adaptors



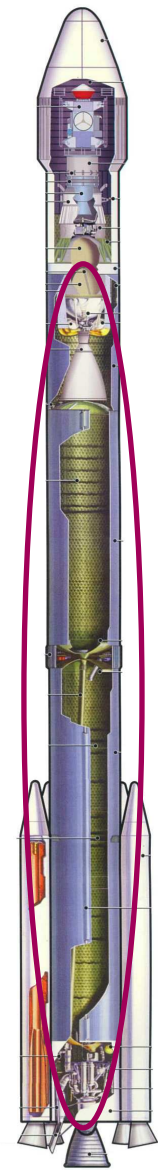
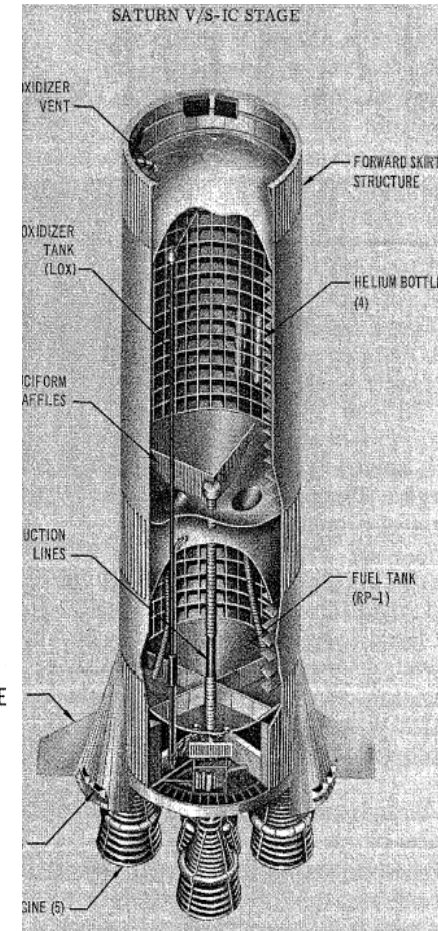
Spacecraft structures

Typical structures

- Spacecraft
- Launch vehicle
 - Fairings
 - Stage structure
 - Thrust structure
 - Adaptors



Saturn V S-II Second Stage Subassemblies

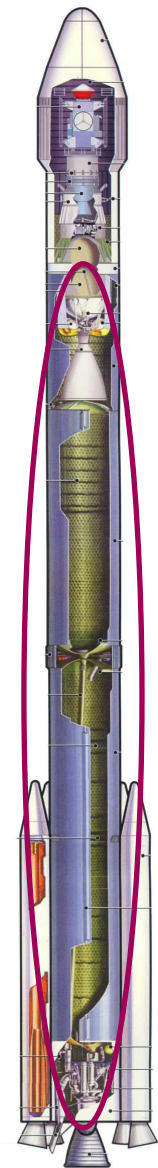
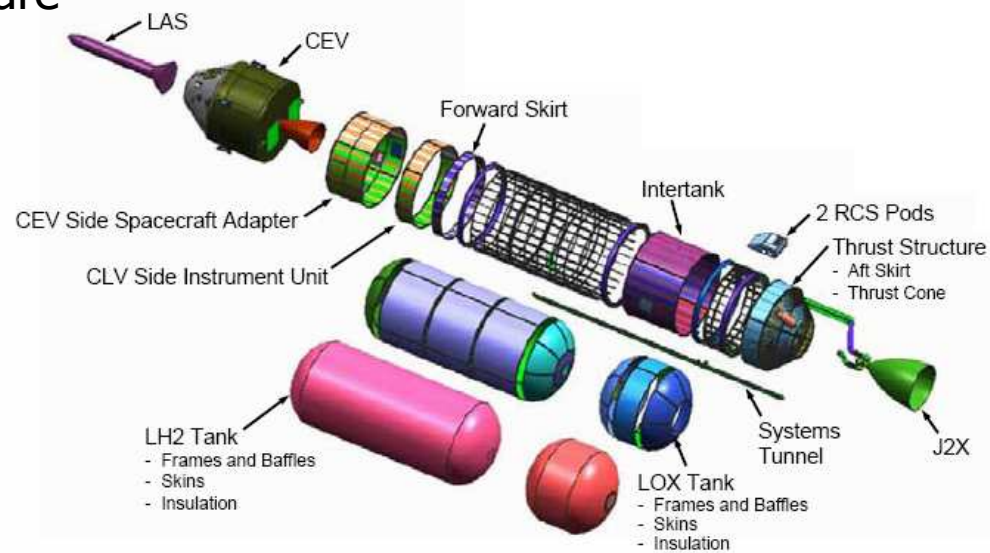


Spacecraft structures

Typical structures

- Spacecraft
- Launch vehicle
 - Fairings
 - Stage structure
 - Thrust structure
 - Adaptors

- Semi-monocoque (load bearing skin internally stiffened)
- External skin, internal tanks separated by longerons and circular stiffeners

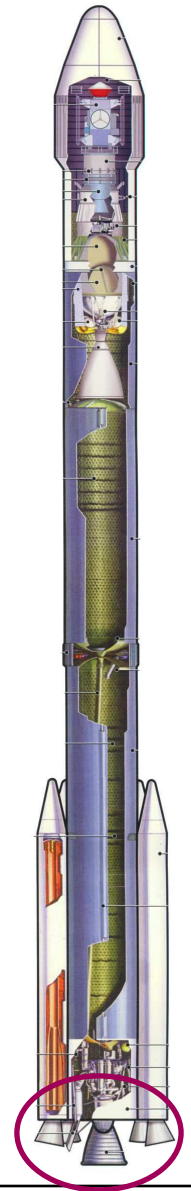
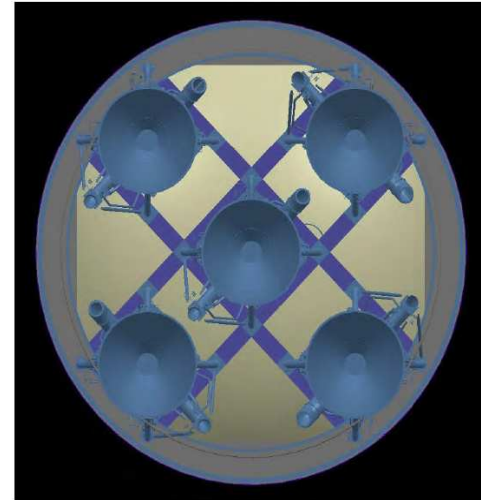
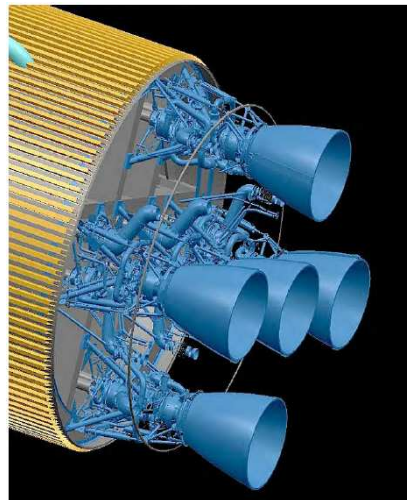


Spacecraft structures

Typical structures

- Spacecraft
- Launch vehicle
 - Fairings
 - Stage structure
 - Thrust structure
 - Adaptors

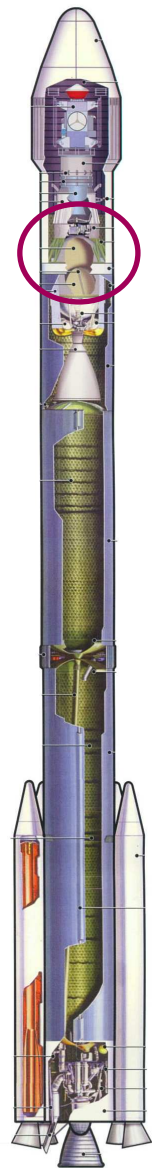
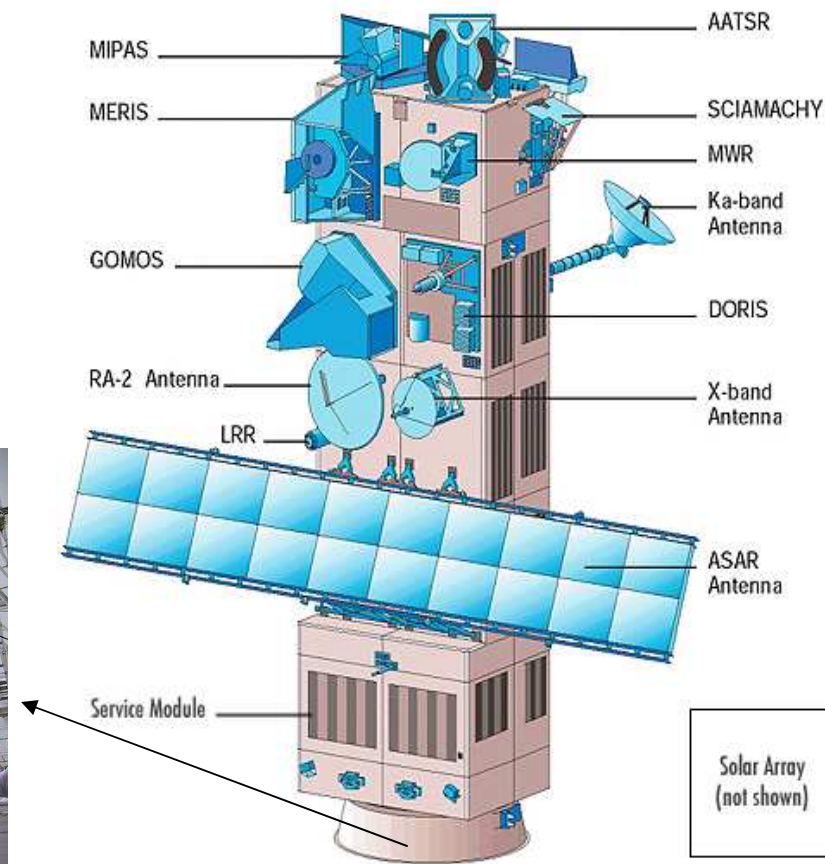
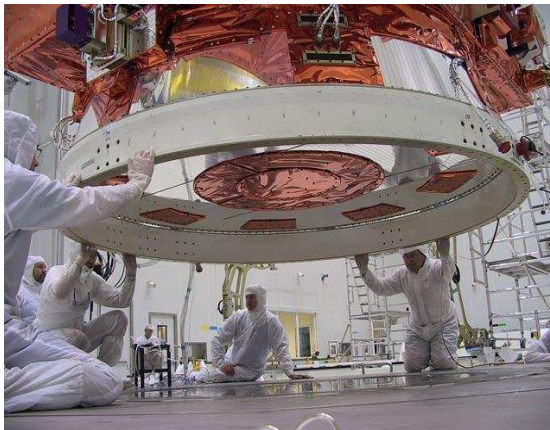
Example: ARIANE 5 ~ 1200 tons of thrust!!



Spacecraft structures

Typical structures

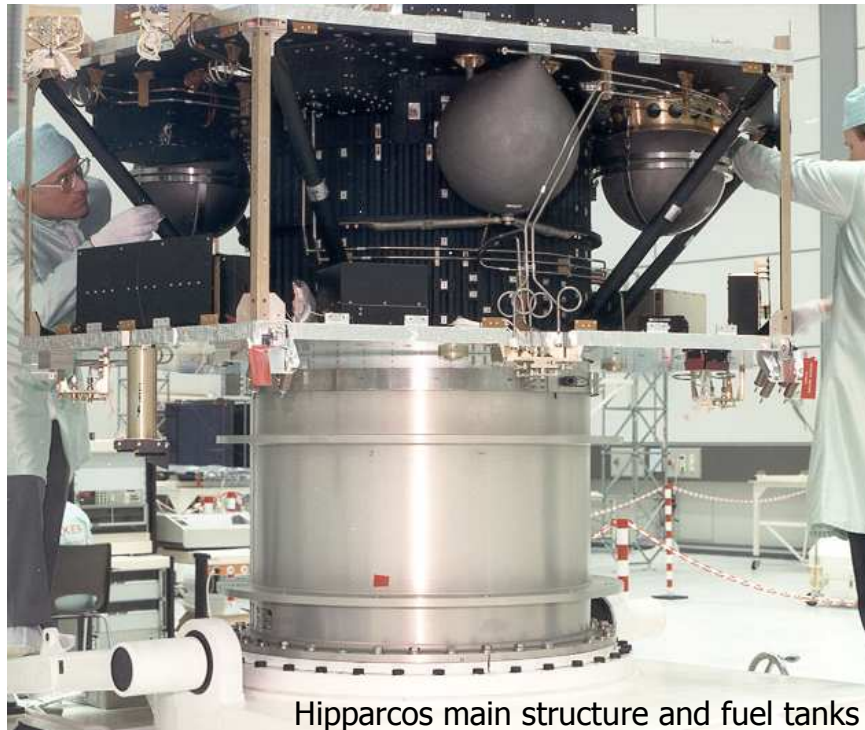
- Spacecraft
- Launch vehicle
 - Fairings
 - Stage structure
 - Thrust structure
 - Adaptors



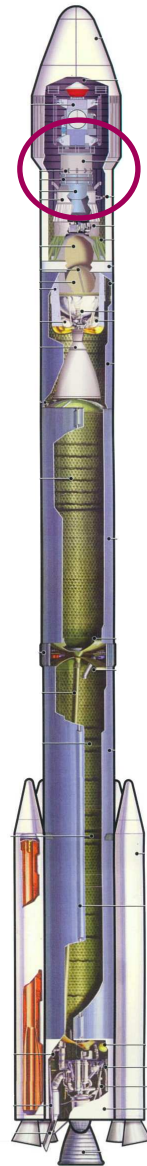
Spacecraft structures

Typical structures

- Spacecraft
- Launch vehicle
 - Fairings
 - Stage structure
 - Thrust structure
 - Adaptors



Hipparcos main structure and fuel tanks



Spacecraft structures

Mission requirements

- Minimum mass
- High stiffness
- Withstand the loads (= high strength)

- Accommodate payload + equipment
- High reliability
- Low cost
- Accessibility, manufacturability

Material selection criteria

Material properties

C. R. corrosion resistance
S. C. C. stress corrosion cracking

Caution: There is considerable variation of the properties of materials according to conditions (ageing, temper, form and structure orientation). Consult manufacturers' data

	Density (kg/m ³)	Young's modulus E(GPa)	Yield strength f(MPa)	Selection criteria				Thermal expansion (μm/m K ⁻¹)	Fracture toughness (MPa m)	Fatigue strength (MPa)	Comment
				E/ρ	E ^{1/2} /ρ	E ^{1/3} /ρ	f _y /ρ				
<i>Aluminium alloy</i>											
6061.T6	2700	68	276	24	2.9	1.5	98.6	23.6	186	97	Good C.R.
7075.T6	2800	71	503	26	3.1	1.5	186.3	23.4	24	159	Prone to SSC in T6 Form
<i>Magnesium alloy</i>											
A2 31B	1700	45	220	26	3.9	2.1	129.4	26			Prone to SCC
ZK 60 A.T5 extrn	1700	45	234	26	3.9	2.1	137.6	26		124	
<i>Titanium alloys</i>											
Ti-6Al-4V (annealed)	4400	110	825	25	2.4	1.1	187.5	9	75	500	
(solution treated and aged)			1035						42	690	
<i>Beryllium alloys</i>											
S 65 A	2000	304	207	151	8.7	3.4	103.5	11.5			Hot pressed sheet
SR 200 E			345								Low fracture toughness
<i>Ferrous alloys</i>											
INVAR		150	275/415					1.66			Low expansion Ferromagnetic
<i>Stainless steel</i>											
AM 350 (SCT850)	7700	200	1034	26	1.84	0.8	134.3	11.9	40/60	550	
304L Ann	7800	193	170	25	1.8	0.7	21.8	17.2			Austenitic
<i>Composites</i>											
KEVLAR 49 0°	1380	76*	1379 [†]	55	6.3	3.1	999.3	-4			Structure members
(Aramid fibre) 90°	1380	5.5	29.6	4	1.7	1.3	21.4	57			Pressure vessels Rocket casings
Graphite epoxy sheets (undirec- tional) GY70/934	1620	282	586	174	10.4	4.0	361.7	-14.7 (Longitudinal) 29.7 (Transverse)			Sheet
Column ref. (see text)	A	B	C	D	E	F	G	H	I	J	

*Tensile modulus

[†]Tensile strength

Material selection criteria

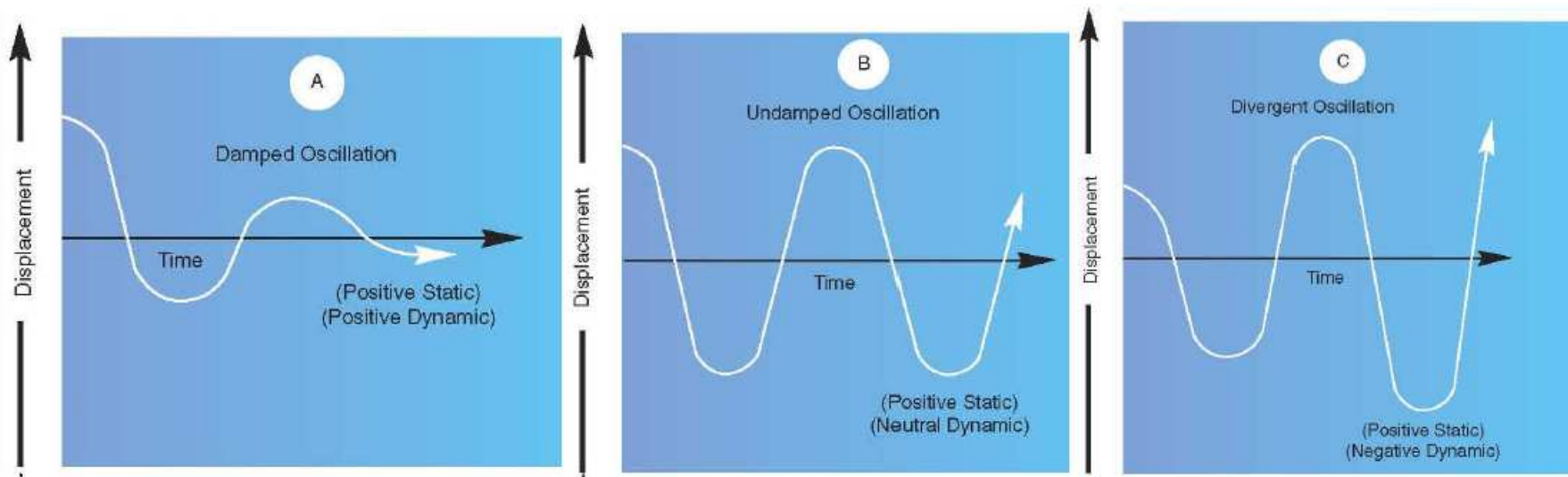
Frequency

- Limiting the natural frequencies of spacecraft is essential to avoid resonance between launch vehicle and spacecraft
- Low dynamic coupling results in lower loads for spacecraft

Material selection criteria

Natural frequency

- Oscillations can be damped or excited
 - Example: aerodynamic flutter



(recall the flutter of horizontal & vertical tail planes)

Material selection criteria

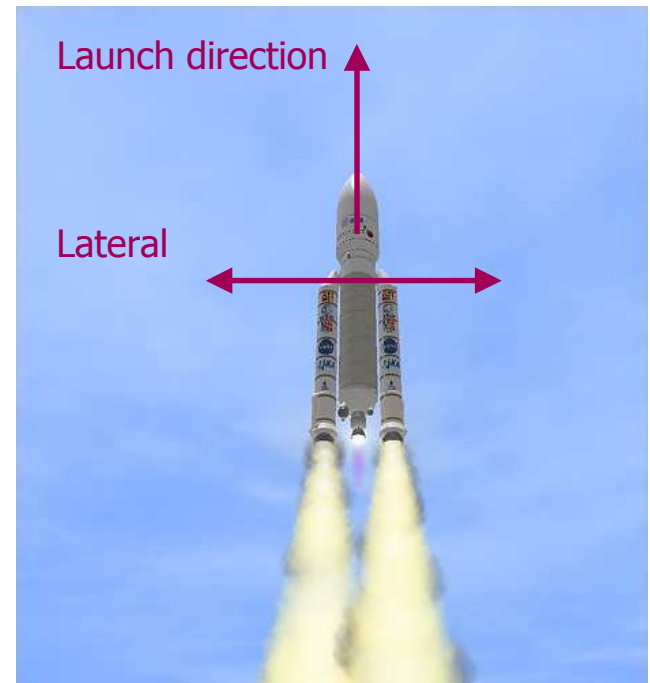
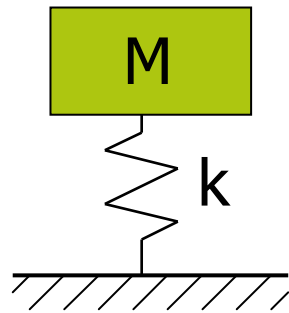
Frequency

- Requirements are launcher dependent:
- **ARIANE 4**
 - Launch direction >31 Hz
 - Lateral >10 Hz
- **STS**
 - Launch direction >13 Hz
 - Lateral >13 Hz

Natural Frequency [Hz]

$$f_n = \frac{1}{2\pi} \sqrt{\frac{k}{m}}$$

k = stiffness = load/deflection



Material selection criteria

Frequency

- Dimensioning the primary structure
- First
 - Lowest natural frequencies $>$ minimum required natural frequencies!
- Then
 - Design for quasi-static loads

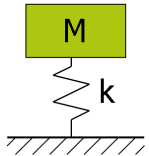
Material selection criteria

Frequency – Example on stiffness sizing

Natural Frequency [Hz]

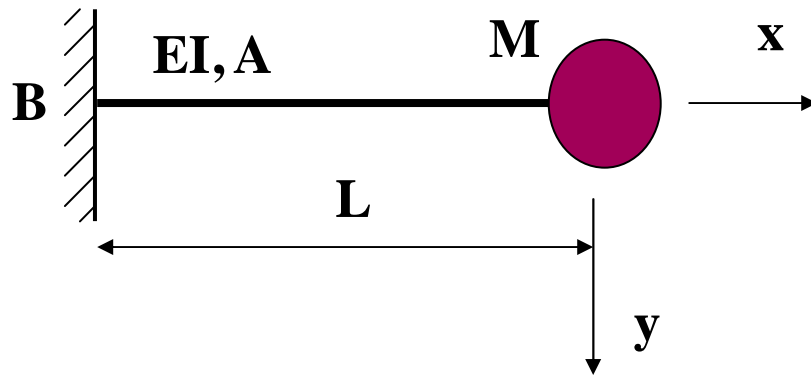
$$f_n = \frac{1}{2\pi} \sqrt{\frac{k}{m}}$$

k = stiffness = load/deflection

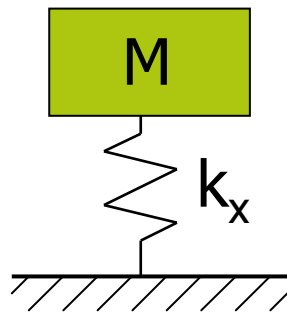


A diagram showing a green rectangular mass labeled 'M' supported by a spring labeled 'k', which is fixed to a hatched ground surface.

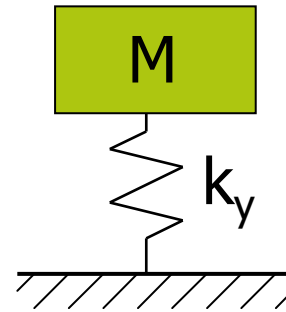
- Mass concentrated at end of clamped beam



- Define single degree of freedom system



$$k_x = \frac{EA}{L}$$



$$k_y = \frac{3EI}{L^3}$$

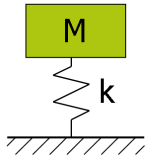
Material selection criteria

Frequency – Example on stiffness sizing

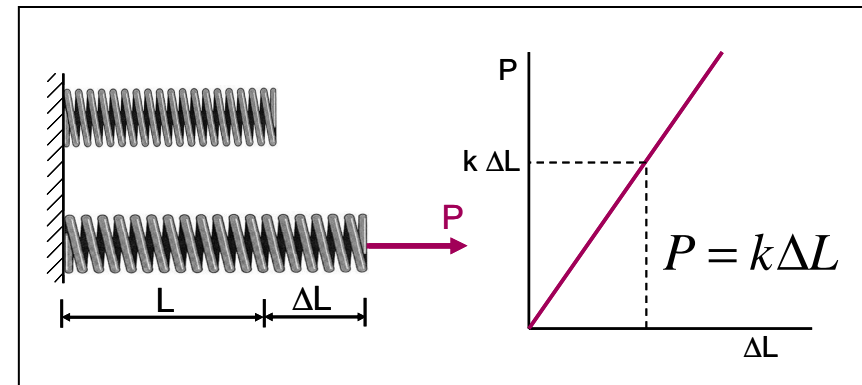
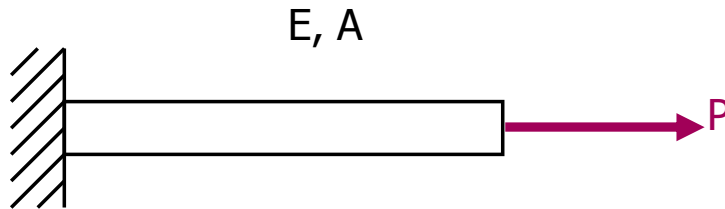
Natural Frequency [Hz]

$$f_n = \frac{1}{2\pi} \sqrt{\frac{k}{m}}$$

k = stiffness = load/deflection



- Axially loaded beam

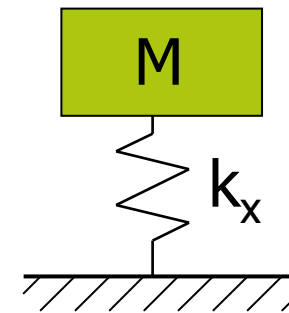


- Strain

$$\varepsilon = \frac{\Delta L}{L} = \frac{P}{EA} = \frac{\sigma}{E}$$

- Elongation

$$\Delta L = P \frac{L}{EA}$$



$$k_x = \frac{EA}{L}$$

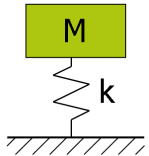
Material selection criteria

Frequency – Example on stiffness sizing

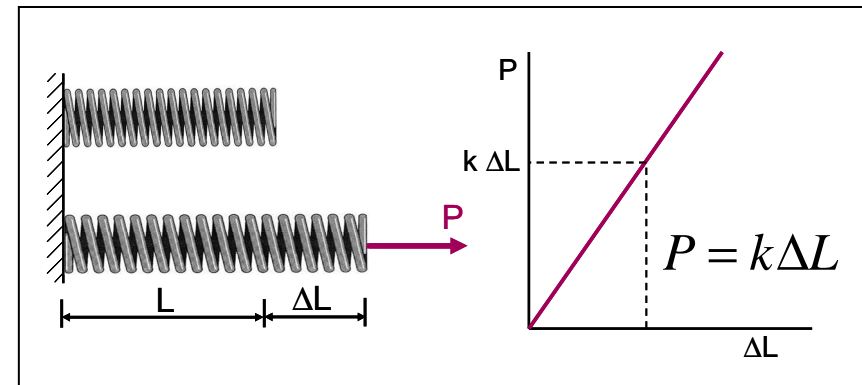
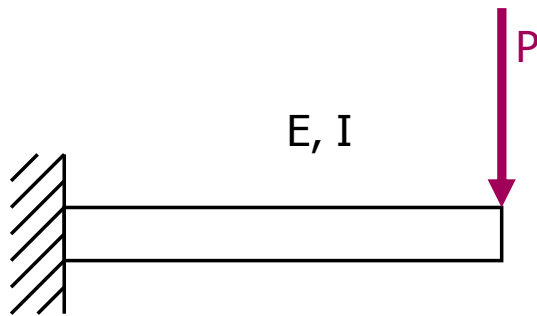
Natural Frequency [Hz]

$$f_n = \frac{1}{2\pi} \sqrt{\frac{k}{m}}$$

k = stiffness = load/deflection

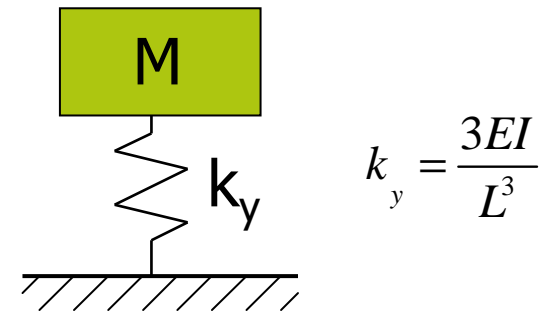


- Laterally loaded beam



- Deflection

$$\delta = \frac{PL^3}{3EI}$$



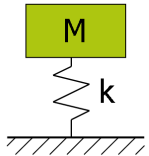
Material selection criteria

Frequency – Example on stiffness sizing

Natural Frequency [Hz]

$$f_n = \frac{1}{2\pi} \sqrt{\frac{k}{m}}$$

k = stiffness = load/deflection



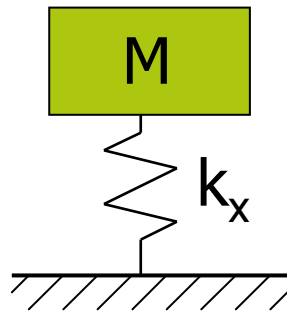
The diagram shows a green rectangular mass labeled 'M' supported by a spring labeled 'k'. The spring is attached to a fixed base, indicated by hatching below it.

- The natural frequency is

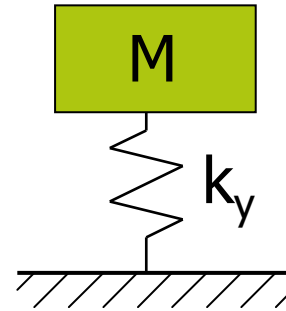
$$f_n = \frac{1}{2\pi} \sqrt{\frac{k}{m}} \quad [Hz]$$

- Which means for the stiffness

$$\frac{3EI}{L^3} \geq (2\pi f_n)^2 M \quad \frac{EA}{L} \geq (2\pi f_n)^2 M$$



$$k_x = \frac{EA}{L}$$



$$k_y = \frac{3EI}{L^3}$$

SMAD III, Fig. 11-42
Equations 11-57,11-58

Material selection criteria

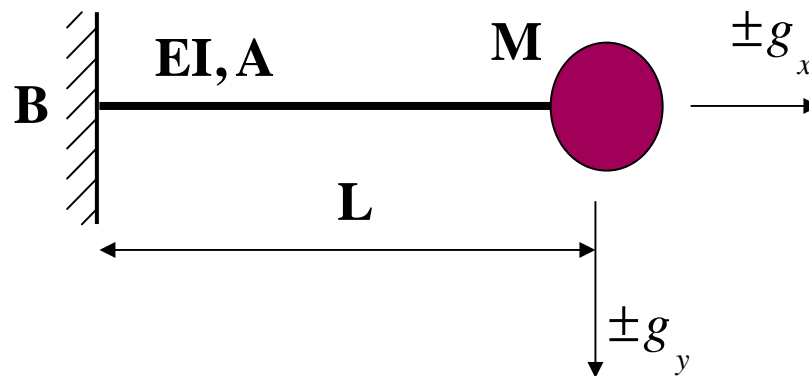
Frequency

- Dimensioning the primary structure
- First
 - Lowest natural frequencies $>$ minimum required natural frequencies!
- Then
 - Design for quasi-static loads

Sizing Example Strength

Frequency – Example on stiffness sizing

- Maximum stress at clamping



Structure/geometry

- Define I and A

Material

- Select material with E

- Axial $\sigma_x = \frac{g_x M}{A} = \frac{F}{A}$

- Lateral $\sigma_y = \frac{g_y M L y}{I} = \frac{F L y}{I}$

$$\sigma_{tot} = \sigma_x + \sigma_y \leq \sigma_{allowable}$$

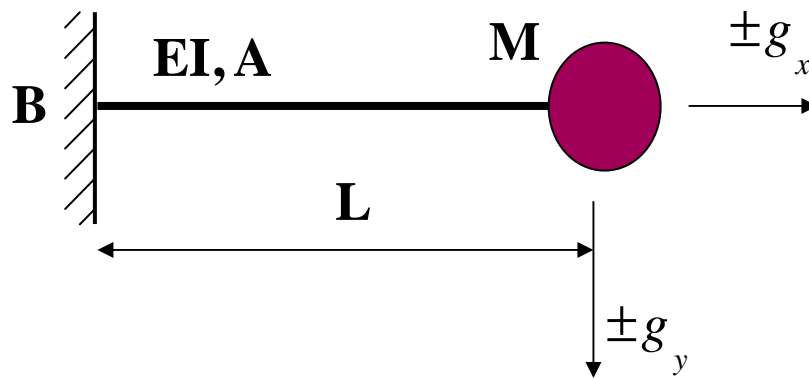
SMAD III

Equations 11-48,11-49

Sizing Example Strength

Frequency – Example on stiffness sizing

- Buckling load



Structure/geometry

- Define I and L

Material

- Select material with E

⇒ Bending stiffness EI !

- Euler
$$F_{euler} = \frac{\pi^2 EI}{4L^2} \leq g_x M$$

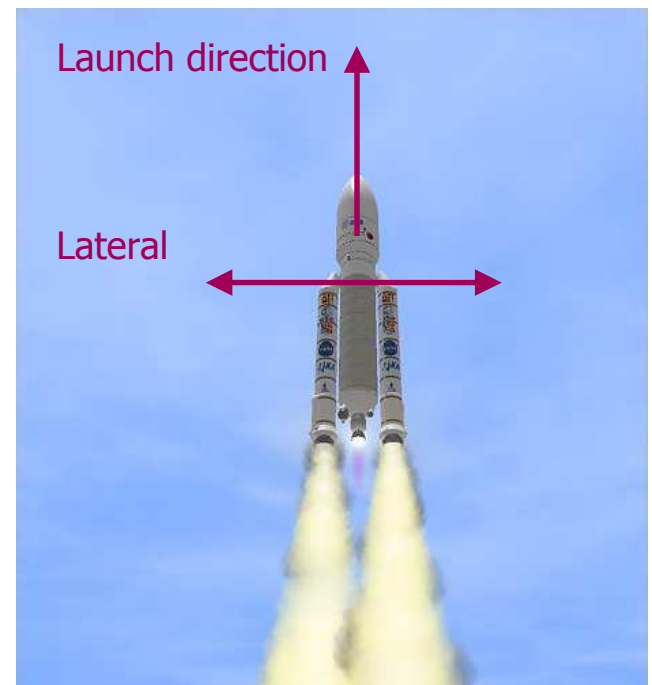
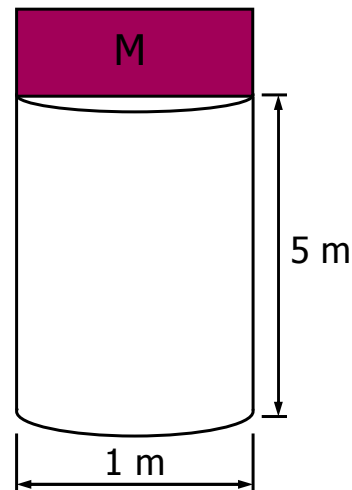
SMAD III

Equations 11-48,11-49

Material selection criteria

Example

- Required wall thickness
- **ARIANE 4**
 - Launch direction >31 Hz
 - Lateral >10 Hz
- Conditions
 - $M=250$ kg
 - $E = 72$ GPa (aluminium)
 - Max. axial acceleration = 6 g (launch load)
 - Payload = point mass
 - $M_{\text{cylinder}} = 0$



Material selection criteria

Example

- Required wall thickness (frequency)
 - Launch direction >31 Hz

$$f_n = \frac{1}{2\pi} \sqrt{\frac{k}{m}} \geq 31 \text{ Hz}$$

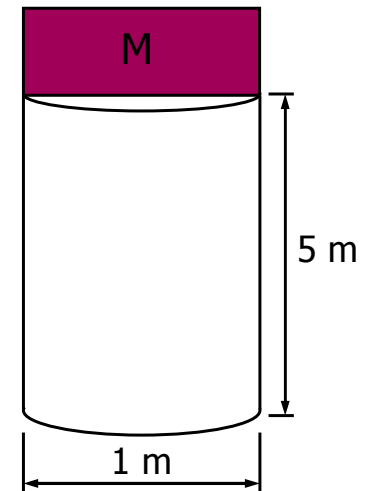
- With $m=250$ kg

$$k = \frac{EA}{L}$$

$$\Rightarrow A \geq \frac{mL}{E} (2\pi f_1)^2$$

$$\text{with } A = 2\pi r t$$

$$\Rightarrow t \sim 1 \text{ mm}$$



Material selection criteria

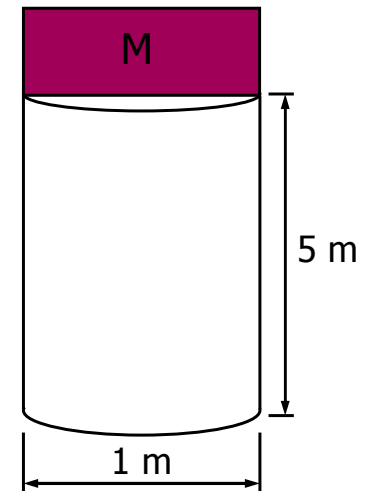
Example

- Required wall thickness (quasi-static loads)
 - Axial acceleration = 6 g
 - Lateral direction = 1.5 g

- Axial $\sigma_x = \frac{g_x M}{A} = \frac{F}{A}$
- Lateral $\sigma_y = \frac{g_y M L y}{I} = \frac{F L y}{I}$

$$\left. \begin{array}{l} \sigma_x = \frac{g_x M}{A} = \frac{F}{A} \\ \sigma_y = \frac{g_y M L y}{I} = \frac{F L y}{I} \end{array} \right\} \sigma_{tot} = \sigma_x + \sigma_y \leq \sigma_{allowable}$$

- With $\sigma_{allowable} = \sigma_{ultimate} / \text{safety factor} \Rightarrow A \ \& \ I$



Summary

Space

- Typical spacecraft & launch vehicle structures
- Steps in dimensioning a spacecraft structure
 - Natural frequency
 - Quasi-static loads
- Example simplified structure for minimum natural frequency requirements