

opgave no.

naam

studienummer

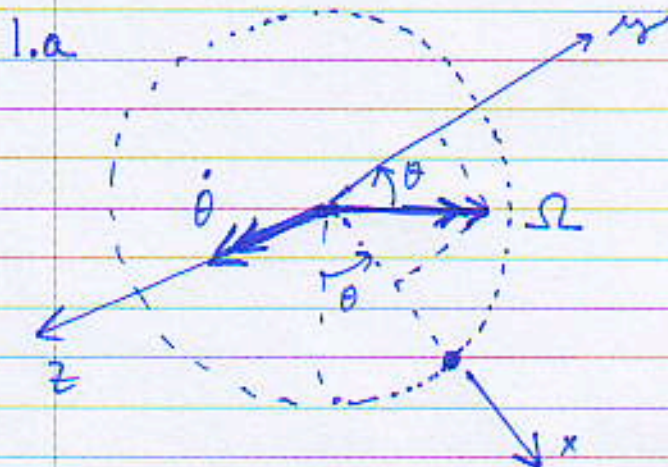
vak AE3-914

code

datum 13-4-6

fac.

Gebruik voor elke opgave een afzonderlijk vel papier!



$$\underline{r}_{\text{rel}} = R \underline{i}$$

$$\underline{\omega} = \Omega \sin \theta \underline{i} + \Omega \cos \theta \underline{j} + \dot{\theta} \underline{k}$$

$$\underline{v}_{\text{rel}} = \underline{0}$$

$$\underline{v}_{\text{cgs}} = \underline{0}$$

$$\underline{v} = \underline{v}_{\text{cgs}} + \underline{\omega} \times \underline{r}_{\text{rel}} + \underline{v}_{\text{rel}} =$$

$$= \begin{vmatrix} \underline{i} & \underline{j} & \underline{k} \\ \Omega \sin \theta & \Omega \cos \theta & \dot{\theta} \\ R & 0 & 0 \end{vmatrix} = R \dot{\theta} \underline{j} - R \Omega \cos \theta \underline{k}$$

$$T = \frac{1}{2} m v^2 = \frac{1}{2} m R^2 (\dot{\theta}^2 + \Omega^2 \cos^2 \theta)$$

$$V = \frac{pQ \cos \theta}{4\pi \epsilon_0 R^2}$$

$$L = T - V$$

$$= \frac{1}{2} m R^2 (\dot{\theta}^2 + \Omega^2 \cos^2 \theta) - \frac{pQ \cos \theta}{4\pi \epsilon_0 R^2}$$

1.b

$$\frac{\partial L}{\partial \dot{\theta}} = m R^2 \dot{\theta}; \quad \frac{d}{dt} \left(\frac{\partial L}{\partial \dot{\theta}} \right) = m R^2 \ddot{\theta}$$

$$\frac{\partial L}{\partial \theta} = -m R^2 \Omega^2 \cos \theta \sin \theta + \frac{P Q \sin \theta}{4 \pi \epsilon_a R^2}$$

$$\frac{d}{dt} \left(\frac{\partial L}{\partial \dot{\theta}} \right) - \frac{\partial L}{\partial \theta} = 0$$

$$m R^2 \ddot{\theta} + m R^2 \Omega^2 \sin \theta \cos \theta - \frac{P Q \sin \theta}{4 \pi \epsilon_a R^2} = 0$$

1.c

$$\ddot{\theta} + \Omega^2 \sin \theta \cos \theta - \frac{P Q \sin \theta}{4 \pi \epsilon_a m R^4} = 0$$

$$\theta = 0 \Rightarrow \ddot{\theta} = 0 \quad (\text{indeed, equilibrium point})$$

Linearisation Method

$$\left. \begin{aligned} \dot{\theta} &= \varphi \\ \dot{\varphi} &= -\Omega^2 \sin \theta \cos \theta + \frac{P Q \sin \theta}{4 \pi \epsilon_a m R^4} \end{aligned} \right\}$$

At $\theta = 0$, linearisation provides:

$$\begin{bmatrix} \dot{\theta} \\ \dot{\varphi} \end{bmatrix} = \begin{bmatrix} 0 \\ -\Omega^2 (\cos^2 \theta - \sin^2 \theta) + \frac{P Q \cos \theta}{4 \pi \epsilon_a m R^4} \end{bmatrix} \Bigg|_{\theta=0} \begin{bmatrix} \theta \\ \varphi \end{bmatrix}$$

$$= \begin{bmatrix} 0 & 1 \\ -\Omega^2 + \frac{PQ}{4\pi\epsilon_0 m R^3} & 0 \end{bmatrix} \begin{bmatrix} \theta \\ \varphi \end{bmatrix}$$

Eigenvalues of Jacobian:

$$\begin{vmatrix} -\lambda & 1 \\ -\Omega^2 + \frac{PQ}{4\pi\epsilon_0 m R^3} & -\lambda \end{vmatrix} = 0$$

$$\lambda^2 = \frac{PQ}{4\pi\epsilon_0 m R^3} - \Omega^2$$

For stability $\lambda \in \mathbb{C} - \mathbb{R}$ or $\text{Re}(\lambda) < 0$.

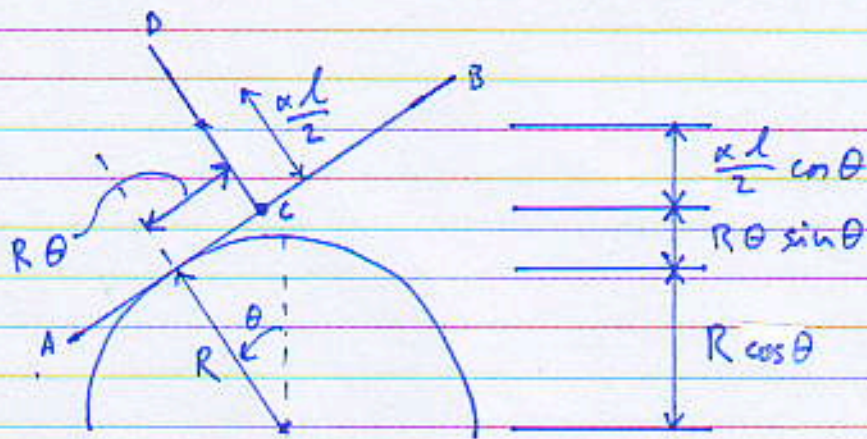
In this case: $\frac{PQ}{4\pi\epsilon_0 m R^3} - \Omega^2 < 0$ for stability

if $Q < 0$, stability condition is always fulfilled

if $Q > 0$, then the condition in Ω is

$$\Omega^2 > \frac{PQ}{4\pi\epsilon_0 m R^3}$$

2.a



$$V_{AB} = m_{AB} g h_{AB} = \rho A l g (R \cos \theta + R\theta \sin \theta)$$

$$V_{CD} = m_{CD} g h_{CD} = \rho A \alpha l g (R \cos \theta + R\theta \sin \theta + \frac{\alpha l}{2} \cos \theta)$$

$$V = V_{AB} + V_{CD} =$$

$$= \rho A l g \left((1+\alpha) R (\cos \theta + \theta \sin \theta) + \frac{\alpha^2 l}{2} \cos \theta \right)$$

2.b $R = 1 \text{ m}$, $A = 4 \cdot 10^{-4} \text{ m}^2$, $\rho = 2500 \text{ kg/m}^3$, $l = 2 \text{ m}$, $g = 10 \frac{\text{m}}{\text{s}^2}$

$$V = 20 \left((1+\alpha + \alpha^2) \cos \theta + (1+\alpha) \theta \sin \theta \right)$$

$$V' = 20 \left(-(1+\alpha + \alpha^2) \sin \theta + (1+\alpha) (\sin \theta + \theta \cos \theta) \right)$$

$$V'(0) = 0 \Rightarrow \theta = 0 \text{ indeed equilibrium}$$

$$V'' = 20 \left(-(1+\alpha + \alpha^2) \cos \theta + (1+\alpha) (\cos \theta + \cos \theta - \theta \sin \theta) \right)$$

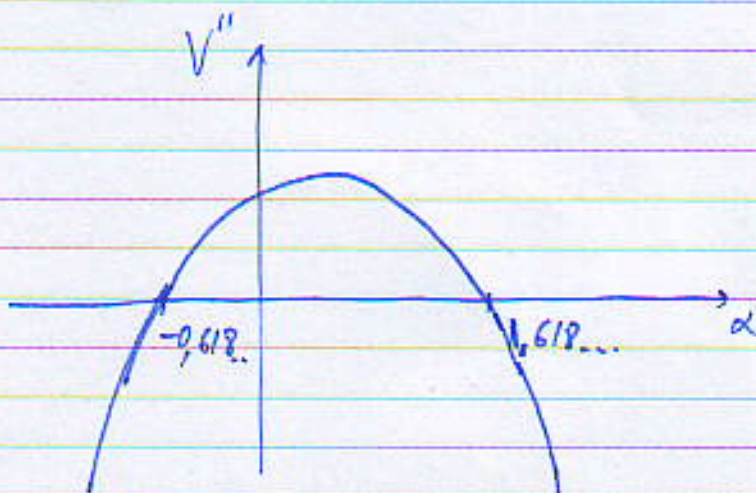
$$V''(0) = 20 \left(-(1+\alpha + \alpha^2) + 2(1+\alpha) \right) = 20(1+\alpha - \alpha^2)$$

$V''(0) > 0$ for stability

$$1 + \alpha - \alpha^2 > 0$$

$$1 + \alpha - \alpha^2 = 0 \Rightarrow \alpha = \frac{-1 \pm \sqrt{1+5}}{-2} = \begin{cases} 1,618\dots \\ -0,618\dots \end{cases}$$

V'' v.s. α :



If $\alpha < 0$ the mass in V_{CG} would be negative, so the only acceptable interval for α is

$$0 \leq \alpha \leq 1,618\dots$$

By the way, 1,618... is the golden section,

$$\Phi = \frac{1 + \sqrt{5}}{2}$$

3.a

$$L = T - V = \frac{1}{2} m (\dot{r}^2 + \dot{s}^2) + \beta(r)\dot{s} + \gamma(r) - V(r)$$

s is ignorable, so

$$\frac{\partial L}{\partial \dot{s}} = \boxed{m\dot{s} + \beta(r)} = C_s \quad \text{is an integral of motion}$$

$L \neq L(t)$ thus

$$h = \frac{\partial L}{\partial \dot{r}} \dot{r} + \frac{\partial L}{\partial \dot{s}} \dot{s} - L =$$

$$m\dot{r}^2 + m\dot{s}^2 + \beta(r)\dot{s} - \frac{1}{2} m (\dot{r}^2 + \dot{s}^2) - \beta(r)\dot{s} - \gamma(r) + V(r)$$

$$= \boxed{\frac{1}{2} m (\dot{r}^2 + \dot{s}^2) - \gamma(r) + V(r)} \quad \text{is an integral of motion}$$

3.b

$$R = C_s \dot{s} - L \quad \dot{s} = \frac{C_s - \beta(r)}{m}$$

$$= C_s \cdot \frac{C_s - \beta(r)}{m} - \frac{1}{2} m \dot{r}^2 - \frac{1}{2} m \left(\frac{C_s - \beta(r)}{m} \right)^2$$

$$- \beta(r) \frac{C_s - \beta(r)}{m} - \gamma(r) + V(r)$$

$$= \boxed{\frac{[C_s - \beta(r)]^2}{2m} - \frac{1}{2} m \dot{r}^2 - \gamma(r) + V(r)}$$

3.c

$$h = R - \frac{\partial R}{\partial \dot{r}} \dot{r}$$

$$= \frac{(C_s - \beta(r))^2}{2m} - \frac{1}{2} m \dot{r}^2 - \gamma(r) + V(r) + m \dot{r}^2$$

$$= \frac{1}{2} m \dot{r}^2 + \underbrace{\frac{(C_s - \beta(r))^2}{2m} - \gamma(r) + V(r)}_{V^{\text{eff}}}$$

$$V^{\text{eff}} = \frac{(C_s - \beta(r))^2}{2m} - \gamma(r) + V(r)$$

4.a

$$\delta M(0) = \delta M(L) = 0$$

$$\boxed{M(0) = 0; \quad M(L) = 0}$$

Essential
boundary
conditions

4.b Ritz's method:

$$\begin{aligned} M(x) &= \alpha h_1(x) + \beta h_2(x) + \gamma h_3(x) \\ &= \alpha + \beta x + \gamma x^2 \end{aligned}$$

Use boundary conditions:

$$M(0) = 0 \Rightarrow \alpha = 0$$

$$M(L) = 0 \Rightarrow \beta = -\gamma L$$

$M(x)$ is reduced to $M(x) = -\gamma L x + \gamma x^2$

Substitute in functional:

$$M'(x) = -\gamma L + 2\gamma x$$

$$M''(x) = 2\gamma$$

$$V(M(x)) = \Phi(\gamma) =$$

$$= \int_0^L \left[\frac{1}{2} E I (2\gamma)^2 - \frac{1}{2} F (2\gamma x - \gamma L)^2 + q(\gamma x^2 - \gamma L x) \right] dx$$

$$= \int_0^L \left[2EIY^2 - \frac{1}{2}F(4Y^2x^2 - 4Y^2xL + Y^2L^2) + q(Yx^2 - YLx) \right] dx$$

$$= 2EIY^2L - \frac{2FY^2L^3}{3} + FY^2L^3 - \frac{1}{2}FY^2L^3 + qY\frac{L^3}{3} - qY\frac{L^3}{2}$$

$$= \left(2EIY^2 - \frac{FY^2L^2}{6} - \frac{qYL^2}{6} \right) L$$

Extremal if:

$$\frac{\partial \phi}{\partial Y} = 0 \Rightarrow$$

$$\left[4EIY - \left(\frac{FY}{3} + \frac{q}{6} \right) L^2 \right] L = 0$$

$$Y \left(4EI - \frac{FL^2}{3} \right) = \frac{qL^2}{6}$$

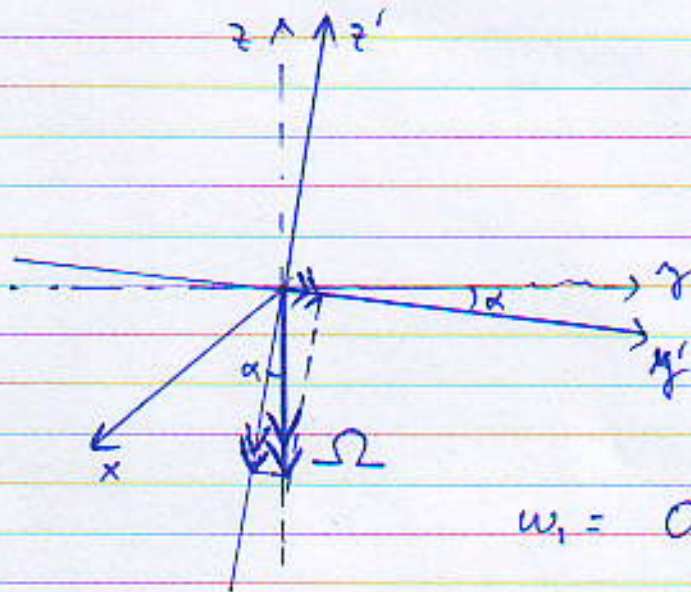
$$Y = \frac{qL^2}{24EI - 2FL^2}$$

$$M(x) = -YLx + Yx^2$$

$$= \frac{qL^2 x(x-L)}{24EI - 2FL^2}$$

5

$$\underline{\underline{I}} = \begin{bmatrix} \frac{1}{4}mr^2 & 0 & 0 \\ 0 & \frac{1}{4}mr^2 & 0 \\ 0 & 0 & \frac{1}{2}mr^2 \end{bmatrix}$$



$$\omega_1 = 0$$

$$\omega_2 = \Omega \sin \alpha$$

$$\omega_3 = -\Omega \cos \alpha$$

$$\dot{\omega}_1 = \dot{\omega}_2 = \dot{\omega}_3 = 0$$

$$M_1 = I_1 \dot{\omega}_1 - (I_2 - I_3) \omega_2 \omega_3 = -\left(-\frac{1}{4}mr^2\right)(-1) \Omega^2 \sin \alpha \cos \alpha$$

$$M_2 = I_2 \dot{\omega}_2 - (I_3 - I_1) \omega_1 \omega_3 = 0$$

$$M_3 = I_3 \dot{\omega}_3 - (I_1 - I_2) \omega_1 \omega_2 = 0$$

Moment components on disc:

$$M_1 = -\frac{1}{8}mr^2 \Omega^2 \sin(2\alpha); \quad M_2 = 0; \quad M_3 = 0$$

Moment components on shaft:

$$M_1 = +\frac{1}{8}mr^2 \Omega^2 \sin(2\alpha); \quad M_2 = 0; \quad M_3 = 0$$