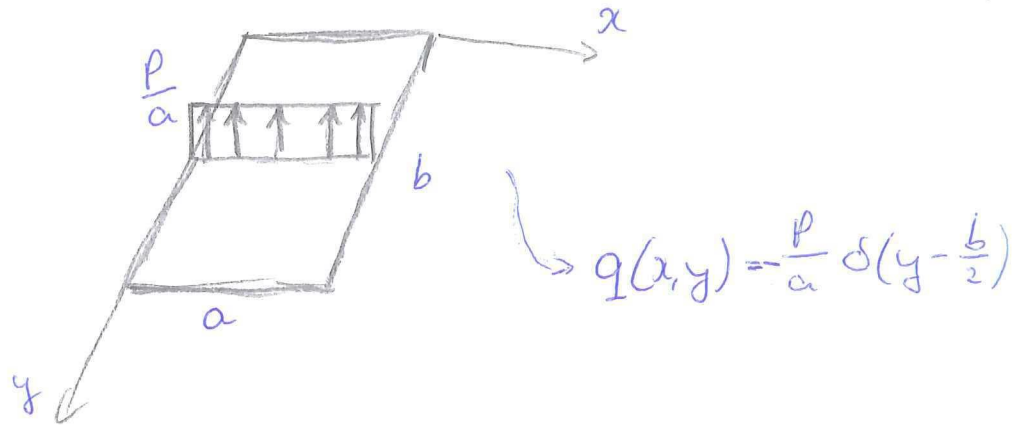


Exercise 2

Problem 1



Governing equation: $w_{xxxx} + 2w_{xyxy} + w_{yyyy} = \frac{q}{D}$ (1)

with $D = \frac{Et^3}{12(1-\nu^2)}$

Assume:

$$w = \sum_{m,n} A_{mn} \sin\left(\frac{m\pi x}{a}\right) \sin\left(\frac{n\pi y}{b}\right) \quad (2)$$

$$q = \sum_{m,n} a_{mn} \sin\left(\frac{m\pi x}{a}\right) \sin\left(\frac{n\pi y}{b}\right) \quad (3)$$

$$(2) \& (3) \rightarrow (1) \Rightarrow A_{mn} = \frac{a_{mn}}{D \left[\left(\frac{m\pi}{a}\right)^2 + \left(\frac{n\pi}{b}\right)^2 \right]^2} \quad (4)$$

Compute a_{mn} :

$$a_{mn} = \frac{4}{ab} \int_0^a \int_0^b q(x,y) \sin\left(\frac{m\pi x}{a}\right) \sin\left(\frac{n\pi y}{b}\right) dy dx \quad (\text{from lecture notes})$$

$$= -\frac{4P}{a^2 b} \int_0^a \sin\left(\frac{m\pi x}{a}\right) dx \int_0^b \sin\left(\frac{n\pi y}{b}\right) \delta\left(y - \frac{b}{2}\right) dy$$

$$= -\frac{4P}{a^2 b} \cdot \frac{a}{m\pi} (1 - (-1)^m) \cdot \sin\left(\frac{n\pi}{2}\right) \quad (\cos(m\pi) = (-1)^m)$$

P.T.O. \downarrow

$$\Rightarrow a_{mn} = \begin{cases} \frac{-\delta P}{ab m \pi} \sin\left(\frac{n\pi}{2}\right) & \text{if } m, n \text{ odd} \\ 0 & \text{otherwise} \end{cases}$$

Substitute into (4) & (2) and use $x = \frac{a}{2}$, $y = \frac{b}{2}$ at center of the plate:

$$w\left(\frac{a}{2}, \frac{b}{2}\right) = -\frac{\delta P}{Dab\pi} \sum_{m,n=1,3,5,\dots} \frac{1}{m} \frac{\sin\left(\frac{m\pi}{2}\right) \sin\left(\frac{n\pi}{2}\right)^2}{\left[\left(\frac{m\pi}{a}\right)^2 + \left(\frac{n\pi}{b}\right)^2\right]^2} = 1$$

$$= -\frac{\delta P}{Dab\pi} \sum_{m,n=1,3,5,\dots} \frac{1}{m} \frac{\sin\left(\frac{m\pi}{2}\right)}{\left[\left(\frac{m\pi}{a}\right)^2 + \left(\frac{n\pi}{b}\right)^2\right]^2}$$

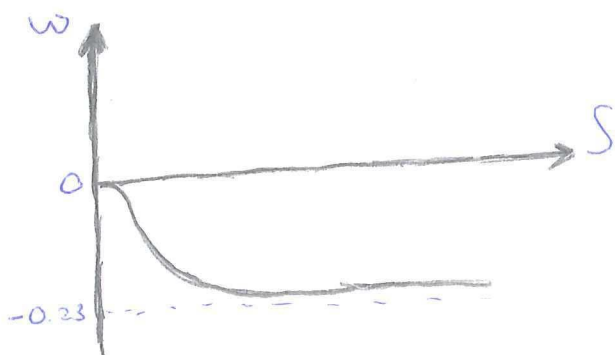
Use aspect ratio $S = \frac{a}{b} \Rightarrow b = \frac{a}{S}$ to give:

$$w_{\text{center}} = \frac{-\delta P S a^2}{D\pi} \sum_{m,n=1,3,5,\dots} \frac{1}{m} \frac{\sin\left(\frac{m\pi}{2}\right)}{\left[(m\pi)^2 + (n\pi S)^2\right]^2}$$

Write out expression for D , and normalize w with respect to $\frac{Pb^3}{Et^3a} = \frac{Pa^2}{Et^3S^3}$:

$$-\frac{\delta P S a^2}{D\pi} \cdot \frac{Et^3 S^3}{Pa^2} = -\frac{\delta Et^3 S^4}{D\pi} = -\frac{gb(1-\nu^2)S^4}{\pi}$$

$$So: w_{\text{center, normalized}} = -\frac{gb(1-\nu^2)S^4}{\pi} \sum_{m,n=1,3,5,\dots} \frac{1}{m} \frac{\sin\left(\frac{m\pi}{2}\right)}{\left[(m\pi)^2 + (n\pi S)^2\right]^2}$$



assuming $\nu = 0.3$

Problem 2

Governing equation for plate deflection =

$$D[w_{xxxx} + 2w_{xyyy} + w_{yyyy}] + N_x w_{xx} + N_y w_{yy} = q$$

NOTE: N_x & N_y assumed positive in compression for convenience

Assume w and q as is (2) and (3) in problem 1.

This gives the buckling condition:

$$D\left[\left(\frac{m\pi}{a}\right)^2 + \left(\frac{n\pi}{b}\right)^2\right]^2 - N_x \left(\frac{m\pi}{a}\right)^2 - N_y \left(\frac{n\pi}{b}\right)^2 = 0$$

Use $a/b = 2$ to get:

$$N_x (m\pi)^2 + 4N_y (n\pi)^2 = \frac{D}{a^2} \left[(m\pi)^2 + 4(n\pi)^2 \right]^2$$

Normalize normal forces wr.t $\frac{D}{a^2}$ to get:

$$\hat{N}_x (m\pi)^2 + 4\hat{N}_y (n\pi)^2 = \left[(m\pi)^2 + 4(n\pi)^2 \right]^2 \quad (5)$$

Compute critical modes:

Case I: \hat{N}_x only, $\hat{N}_y = 0$

$$\text{into (5)} \rightarrow \hat{N}_x = \frac{1}{(m\pi)^2} \left[(m\pi)^2 + 4(n\pi)^2 \right]^2 = \left[m\pi + \frac{4n^2\pi}{m} \right]^2$$

\Rightarrow minimized for $m=2, n=1$

$$\text{set } m=2, n=1 \text{ in (5)} \rightarrow \hat{N}_x \cdot 4\pi^2 + \hat{N}_y \cdot 4\pi^2 = (8\pi^2)^2$$

$$\boxed{\hat{N}_y = 16\pi^2 - \hat{N}_x} \quad (6)$$

Case II: \hat{N}_y only, $\hat{N}_x = 0$

$$\text{into (5)} \rightarrow \hat{N}_y = \frac{1}{4(n\pi)^2} \left[(m\pi)^2 + 4(n\pi)^2 \right]^2 = \left[\frac{m^2\pi}{2n} + 2n\pi \right]^2$$

\Rightarrow minimized for $m=1, n=1$

$$\text{set } m=1, n=1 \text{ in (5)} \rightarrow \hat{N}_x \pi^2 + \hat{N}_y \cdot 4\pi^2 = (5\pi^2)^2$$

$$\boxed{\hat{N}_y = \frac{25\pi^2 - \hat{N}_x}{4}} \quad (7)$$

Plot eq (6) & (7) to obtain buckling interaction curve

curve:

