

Stroom- en Regeltechniek II

Lecture 6 – Root Locus, Frequency Response

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Lecture Outline

Previous lecture: The root locus method, analysis, design.

Today:

- Remarks on the computer session.
- RL: additional examples.
- Realistic PID controller.
- Frequency response.

Matlab / Simulink Computer Session

- Do your homework:
 - Read the entire handout (know what to do).
 - Work out by hand items a) through e) of Section 5.
- Bring the handout with you to the computer lab.
- Be on time, please, 10 min in advance.

You may bring your own laptop to the lab, if you want.

RL: Effect of Parameter Change

Consider our DC motor in a position control loop:

1. Use root locus to design a P controller for the nominal system with:

$$K_t = 0.5, R = 1, L = 0, b = 0.1, J = 0.01$$

2. Use root locus to analyze how the closed-loop poles change when the moment of inertia J of the load changes.

DC Motor: Root Locus for P Control

$$G(s) = \frac{\theta(s)}{E(s)} = \frac{K_t}{s[(Ls + R)(Js + b) + K_t^2]}$$

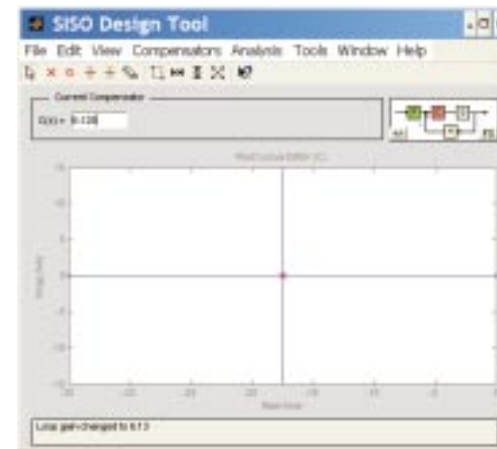
Consider $L = 0$:

$$G(s) = \frac{\theta(s)}{E(s)} = \frac{K_t}{s[(JR s + bR) + K_t^2]} = \frac{k}{s(s + a)}$$

with $k = \frac{K_t}{JR}$, $a = \frac{bR + K_t^2}{JR}$

Design a proportional controller such that the closed loop has a double real pole (use `r1tool` in Matlab).

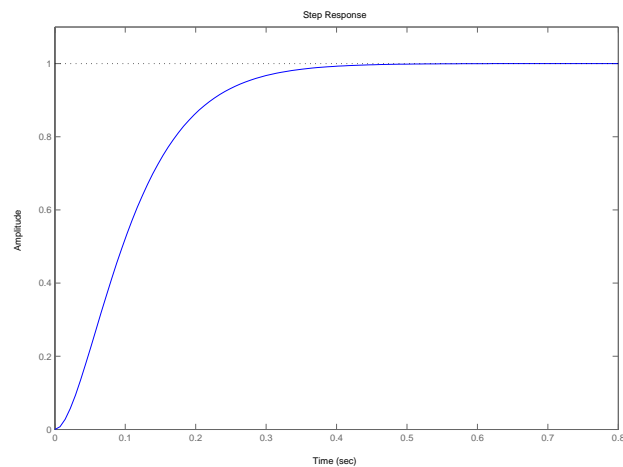
DC Motor: Root Locus for P Control



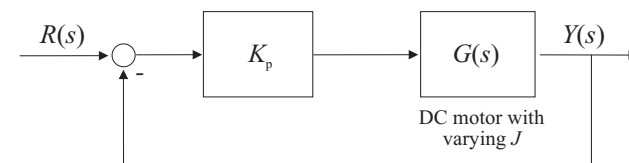
$$K_p = 6.125$$

$$p_{1,2} = -17.5$$

DC Motor: Closed-Loop Step Response



RL for Analysis: Varying Moment of Inertia



$$L(s) = G(s)K_p \quad \text{process and fixed controller in series}$$

K will represent influence of varying moment of inertia J

RL for Analysis: Varying Moment of Inertia

$$G(s)K_p = \frac{k_n}{s(s + a_n)} \quad \text{for the given controller gain } K_p:$$

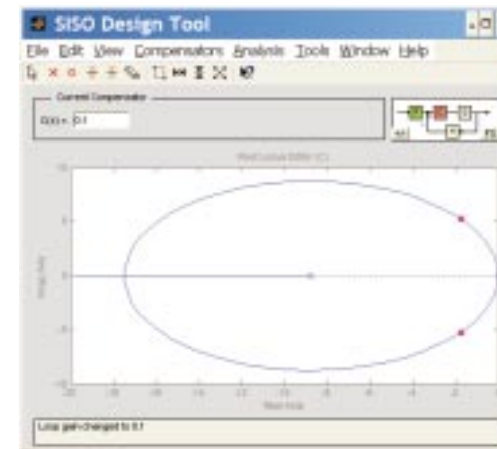
$$k_n = \frac{K_t K_p}{J_n R}, \quad a_n = \frac{bR + K_t^2}{J_n R} \quad \text{with } J_n = 0.01$$

Dividing J_n by factor K means multiplying a_n and k_n by K .

Characteristic equation:

$$s^2 + K(sa_n + k_n) = 0 \quad \Rightarrow \quad 1 + K \frac{sa_n + k_n}{s^2} = 0$$

DC Motor: Root Locus for Varying J

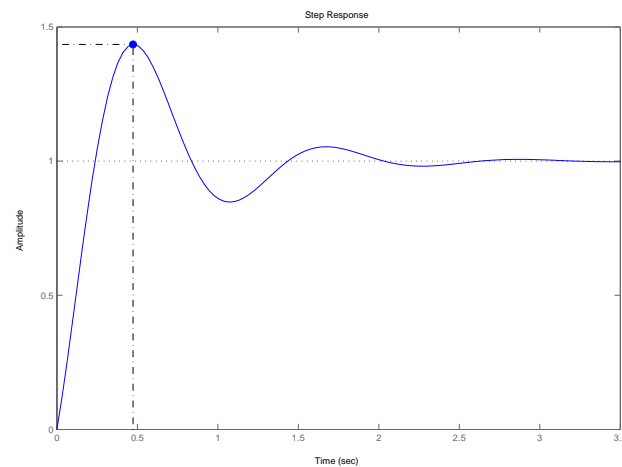


$$K = 0.1$$

↓

$$J = 10 \cdot J_n$$

Large Inertia: Closed-Loop Step Response



Proper Systems

A system $G(s) = \frac{B(s)}{A(s)}$ for which $\deg A(s) \geq \deg B(s)$

is called proper (has not more zeros than poles).

In reality only proper systems exist!

Consequence:

the 'textbook' form of the PD controller cannot be realized:

$$C(s) = K_p(1 + T_d s)$$

More Realistic PD Controller

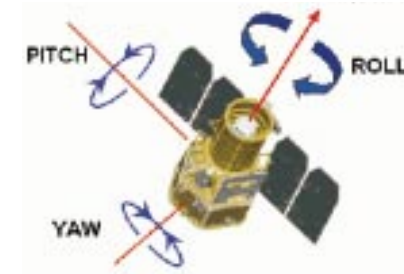
Filtered derivative:

$$C(s) = K_p \left(1 + \frac{T_d s}{(T_d/N)s + 1} \right)$$

where N is typically in the range 10 – 20.

This means that an additional pole is introduced far left on the real axis.

Satellite Attitude Control Revisited



$$\text{Transfer function: } G(s) = \frac{\Theta(s)}{T(s)} = \frac{1}{s^2}$$

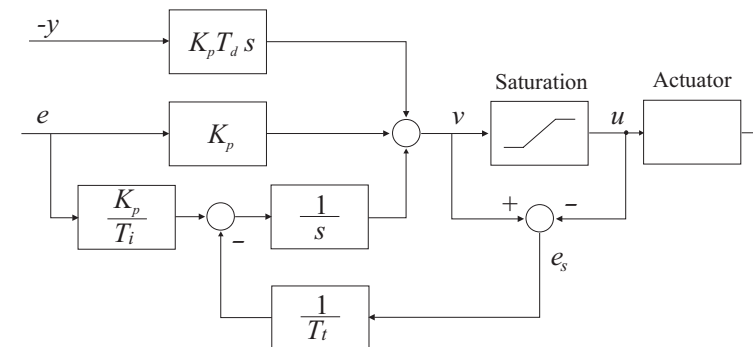
Compare the RL for ideal and realistic PD controller.

PID Controller Used in Practice

$$U(s) = K_p \left(E(s) + \frac{1}{sT_i} E(s) - \frac{T_d s}{(T_d/N)s + 1} Y(s) \right)$$

- Derivative action applied to $-Y(s)$ instead of $E(s)$.
- Anti-windup scheme used for the integral action (prevent integration when the actuator becomes saturated).

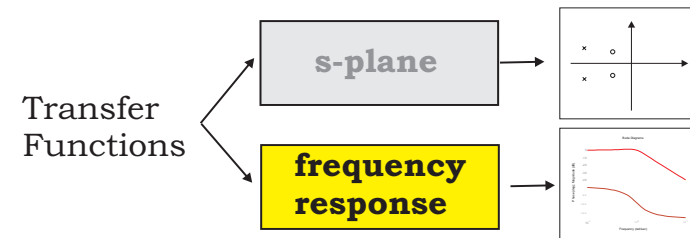
Anti-Windup Tracking Scheme



Root Locus: Homework Assignments

- Read Chapter 5 of the book by Franklin et al.
- Sketch root loci of first, second and third-order systems with real zeros and both real and complex poles by hand.
- For Examples 5.1 through 5.8 in the book verify the results by using Matlab.
- Problems at the end of Chapter 5: work out problems 5.1 and 5.2 by hand and a selection of the remaining problems by using Matlab.

Representations of Transfer Functions

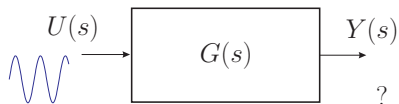


Frequency response:

- Bode plot
- Nyquist plot

Frequency Response: Setting

Consider a linear time invariant system:



Input: $u(t) = M \sin \omega t$

What is the steady-state output?

Frequency Response: Laplace Transform

$$u(t) = M \sin \omega t$$

$$U(s) = \mathcal{L}\{u(t)\} = \frac{M\omega}{s^2 + \omega^2} = \frac{M\omega}{(s + j\omega)(s - j\omega)}$$

$$Y(s) = G(s)U(s) = G(s) \frac{M\omega}{(s + j\omega)(s - j\omega)}$$

$$y(t) = ?$$

Partial Fraction Expansion

General Laplace transform of the output:

$$Y(s) = \sum_{i=1}^n \sum_{j=1}^{m_i} \frac{K_{ij}}{(s-p_i)^j} + \frac{K}{s-j\omega} + \frac{K^*}{s+j\omega}$$

Corresponding time signal:

$$y(t) = \underbrace{\sum_{i=1}^n \sum_{j=1}^{m_i} K_{ij} t^j e^{p_i t}}_{\text{transient}} + \underbrace{K e^{j\omega t} + K^* e^{-j\omega t}}_{\text{periodic signal}}$$

Compute Coefficients K and K*

In steady state:

$$Y(s) = \frac{K}{s-j\omega} + \frac{K^*}{s+j\omega}$$

$$K = Y(s)(s-j\omega)|_{s=j\omega}$$

$$= G(s) \frac{M\omega}{(s+j\omega)(s-j\omega)} (s-j\omega) \Big|_{s=j\omega}$$

$$= G(s) \frac{M\omega}{s+j\omega} \Big|_{s=j\omega} = G(j\omega) \frac{M}{2j}$$

Compute Coefficients K and K*

$$K = \frac{M}{2j} G(j\omega) = \frac{M}{2j} |G(j\omega)| e^{j\angle G(j\omega)}$$

and

$$K^* = -\frac{M}{2j} G(-j\omega) = -\frac{M}{2j} |G(j\omega)| e^{-j\angle G(j\omega)}$$

$$\begin{aligned} K e^{j\omega t} + K^* e^{-j\omega t} &= \\ M |G(j\omega)| \frac{e^{j(\omega t + \angle G(j\omega))} - e^{-j(\omega t + \angle G(j\omega))}}{2j} &= \\ = M |G(j\omega)| \sin(\omega t + \angle G(j\omega)) & \end{aligned}$$

Frequency Response: Summary

$$u(t) = M \sin \omega t$$

$$y(t) \simeq M |G(j\omega)| \sin(\omega t + \angle G(j\omega))$$

$|G(j\omega)|$... magnitude (gain)

$\angle G(j\omega)$... phase