

**Bode** plot

Nyquist plot

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transfer function

data

(experiment)

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frequency response

#### **Frequency Domain Methods**

The key of frequency domain design:

provide sufficient phase at the crossover frequency

(= get the closed-loop far enough from the point of becoming unstable)

 $\Rightarrow$  Bode plots are well suited as a design and analysis tool.

..., so, do we need yet another kind plot?

In fact, we do, let's have a look why ...



# Complex Numbers as Vectors



# **Deficiency of Bode Plots**

For systems with poles in right half-plane, the Bode plot alone does not provide any good indication of stability / instability.

 $\rightarrow$  In the above example, the phase will never cross  $-180^\circ,$  and yet, for K<0.1, the closed loop becomes unstable (check the root locus!).

Is there a method for frequency domain design, considering stability for all kinds of systems?

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6

Yes, the Nyquist plot and stability criterion.

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### **Argument Principle for Stability Analysis**

Given the Nyquist plot of KG(s), we want to determine whether the closed loop:

$$G_{cl}(s) = \frac{Y(s)}{R(s)} = \frac{KG(s)}{1 + KG(s)}$$

is stable.

- Closed-loop stability  $\iff G_{cl}(s)$  has no poles in RHP.
- Poles are given by 1 + KG(s), so let us study 1 + KG(s).

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15

#### **Argument Principle in General**

- For a clockwise contour in the *s*-plane, denote:
- P number of poles encircled in the s-plane
- ${\it Z}$  number of zeros encircled in the  ${\it s}\mbox{-}{\it plane}$
- N number of clockwise encirclements of the origin by G(s)

$$\mathsf{N}=\mathsf{Z}$$
 -  $\mathsf{P}$ 

Recall:

$$\angle G(s) = \sum_{i} \angle (s - z_i) - \sum_{j} \angle (s - p_j) = \sum_{i} \psi_i - \sum_{j} \phi_j$$
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#### **Argument Principle for Stability Analysis**

$$G_{cl}(s) = \frac{Y(s)}{R(s)} = \frac{KG(s)}{1+KG(s)}$$

Poles of  $G_{cl}(s)$  are the solutions of 1 + KG(s) = 0, i.e.:

poles of  $G_{cl}(\boldsymbol{s})$  are the zeros of  $(1+KG(\boldsymbol{s}))$ 

in addition, as:

$$1 + KG(s) = 0 \quad \rightarrow \quad 1 + K\frac{b(s)}{a(s)} = 0 \quad \rightarrow \quad \frac{a(s) + Kb(s)}{a(s)} = 0$$

poles of G(s) are the poles of (1 + KG(s))



# **Argument Principle for Stability Analysis**

 $\mathsf{N}=\mathsf{Z}$  -  $\mathsf{P}$ 

Z = number of RHP zeros of (1 + KG(s))P = number of RHP poles of (1 + KG(s))

Given that:

$$1+KG(s)=0 \quad \rightarrow \quad \frac{a(s)+Kb(s)}{a(s)}=0$$

we have:

Z = number of RHP poles of  $G_{cl}(s)$   $\ldots$  CL poles P = number of RHP poles of G(s)  $\ldots$  OL poles







## Nyquist Stability Criterion

$$\mathsf{Z}=\mathsf{N}+\mathsf{P}$$

In words:

number of RHP closed-loop poles =

clockwise encirclements + number of RHP open-loop poles

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# Simple Bicycle Models

Front steering:

$$G(s) = \frac{\phi(s)}{\delta(s)} = K \frac{s + \frac{v}{a}}{s^2 - \frac{g}{h}}$$

Rear steering:

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$$G(s) = \frac{\phi(s)}{\delta(s)} = K \frac{-s + \frac{v}{a}}{s^2 - \frac{g}{h}}$$

 $\boldsymbol{a} - \text{distance}$  of COM to fixed wheel center

h – height of COM above ground

0.....

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23

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# Front Steering: Nyquist Plot

22



# Nyquist: Homework Assignments

- Read Section 6.3 (Nyquist stability criterion).
- Work out examples in this section and verify the results by using Matlab.
- Work out problems 6.18 6.22 and verify your results by using Matlab.

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26