wb1224 - Thermodynamics 2 Lecture 11 - Energy Conversion Systems

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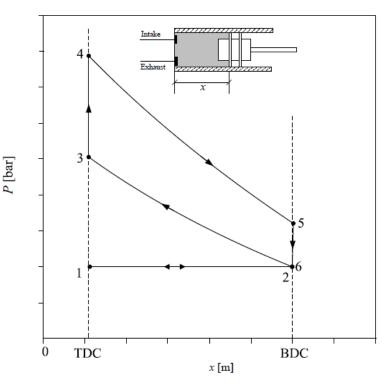
Content Lecture 3 - overview

- Other gas cycles: Overview of the Otto and Diesel cycle (recip engines)
- Comparison Otto, Diesel, Brayton
- Outlook on fuel cells
- Improvement on efficiency: combined cycles and cogeneration
- Exergy: an intro



Ideal Otto cycle (1)

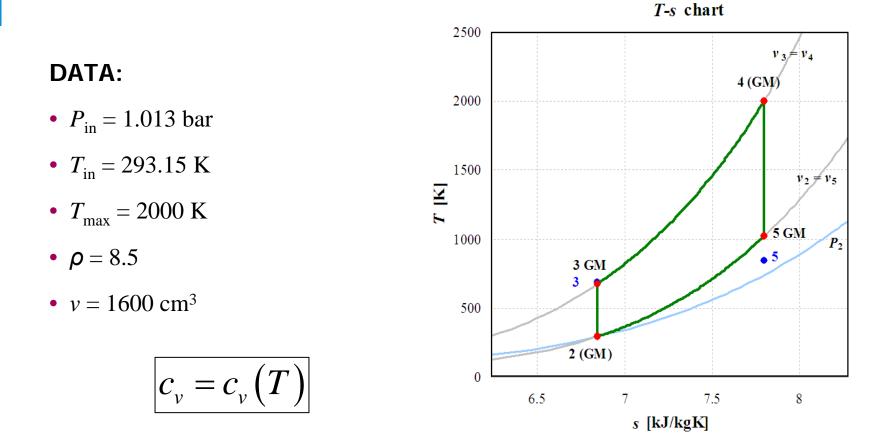
- Open cycle, air as ideal gas (no combustion)
- Model for spark-ignition recip engines
- Valves: top dead center (TDC) and bottom dead center (BDC)
- Ideal: instantaneous ignition at TDC
- Ideal: compression and expansion as isentropic
- Work done on and by the piston only during expansion and compression
- Isochoric combustion and discharge





Ideal Otto cycle (2)

TUDelft



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Ideal Otto cycle (3)

 Assumption: no combustion (no change of working fluid)

- Working fluid: polytropic ideal gas
- Similarly with ideal Brayton cycle:

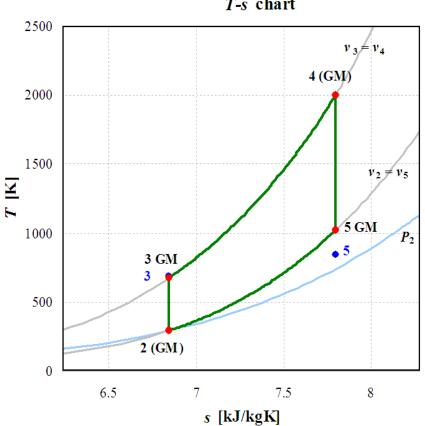
$$Pv = RT, \quad \overline{c_v} = \text{const.}$$

$$w_{\text{comp/exp}} = \Delta u \quad (\text{isochoric})$$

$$\rho \equiv \frac{v_2}{v_3} = \frac{v_5}{v_4} \quad \frac{T_3}{T_2} = \left(\frac{v_3}{v_2}\right)^{\gamma-1}$$

$$\eta = \frac{w_{\text{net}}}{q_{\text{in}}} = 1 - \frac{\overline{c_v} \left(T_5 - T_2\right)}{\overline{c_v} \left(T_4 - T_3\right)} = 1 - \rho^{1-\gamma}$$

TUDelft



T-s chart

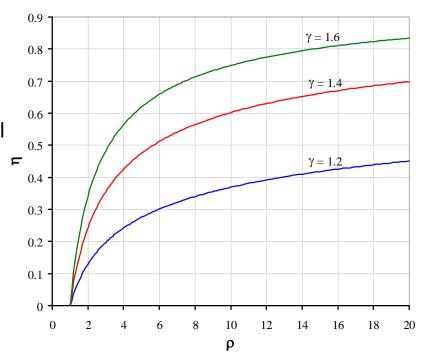
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Efficiency of the ideal Otto cycle

• Difference with Brayton: fluid dynamic

losses $\propto \frac{1}{2} w_{\text{Fluid}}^2$

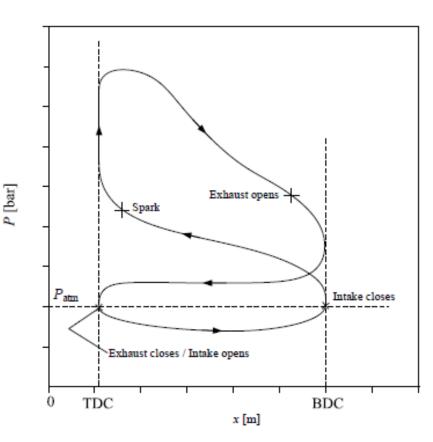
- No maximum value for η as Brayton (real cycle), but η increases with ρ → Ideal cycle efficiency is a good parameter for real-cycle evaluation
- Influence of γ: better if high air/to fuel ratio (more air)





Real Otto cycle

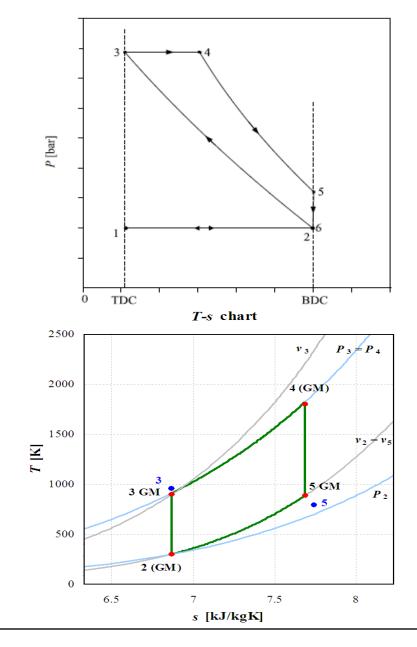
- Combustion takes time: anticipation wtr opening of intake valve/ closing of exhaust valve
- Piston work to push the gases out
 -> gas work on cylinder during intake
- Compression and expansion non adiabatic (non isentropic)
- High ho but problem of detonation





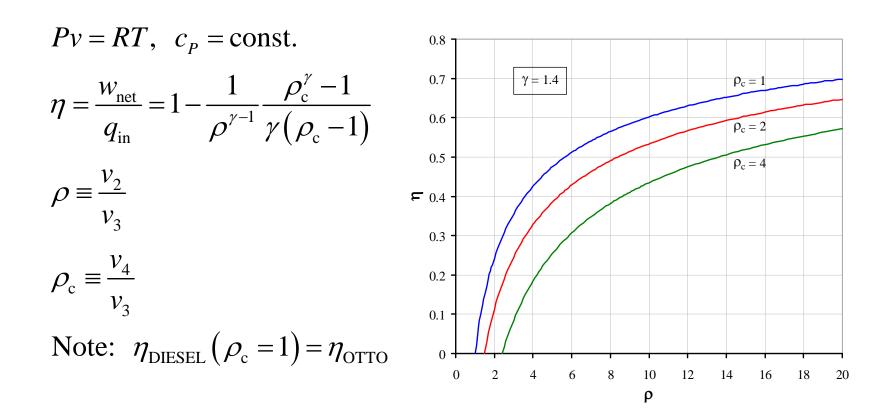
Ideal Diesel cycle

- To avoid detonation → fuel injection after compression/spontaneous ignition
- Isobaric expansion
- Isentropic expansion/compression





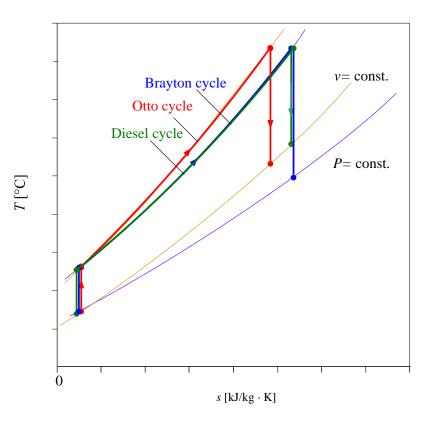
Efficiency of the ideal Diesel cycle



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Comparison Brayton, Otto, Diesel

- If $q_{\rm INPUT}$ is the same: Diesel gives less net work output
- Therefore $\eta_{\rm DIESEL}$ is lower for the same β (ho) and $T_{\rm min}$ and $T_{\rm max}$
- But $eta_{ ext{DIESEL}}$ can be much higher than $eta_{ ext{OTTO}}$ and $eta_{ ext{BRAYTON}}$

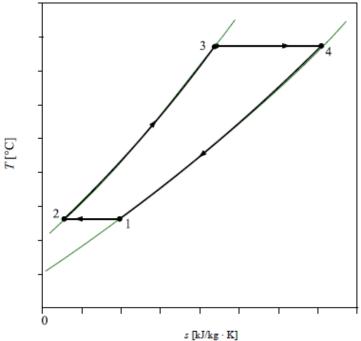




Stirling engine

- Closed cycle
- Isochoric heating and cooling
- Isothermal compression and expansion
- Complex
- Also reversed (cooling-heat pump)

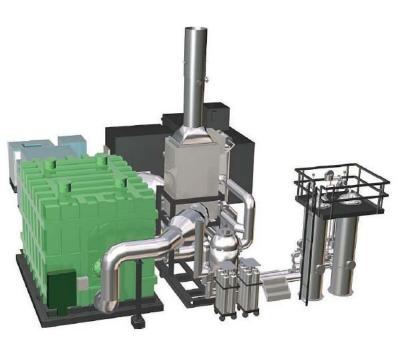






Fuel Cells

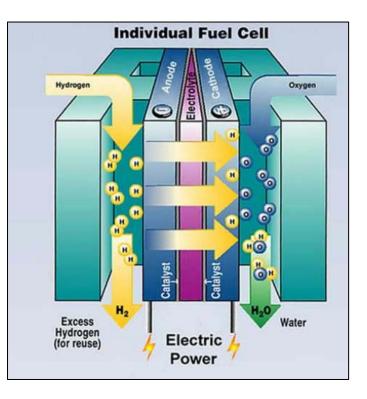
- Direct conversion of fuel into electricity with no thermodynamic cycle: efficiency very high
- Controlled reaction between fuel and oxidizer to produce electricity
- Continuous feed of fuel and oxidizer





Fuel cell: principle of operation

- Porous electrodes between O₂ and H₂
- Electrolyte (solid or liquid) limits reaction rate
- H₂ diffuses through anode, is absorbed on surface of cathode and reacts with OH⁻ in the electrolyte and forms H₂O, yielding free electrons
- O₂ diffuses through the cathode, is absorbed by the surface, and reacts with H₂O forming OH⁻
- H₂O is formed at the anode and decomposed at the cathode,
- Electrons are produced at the cathode and provide electrical current for loads

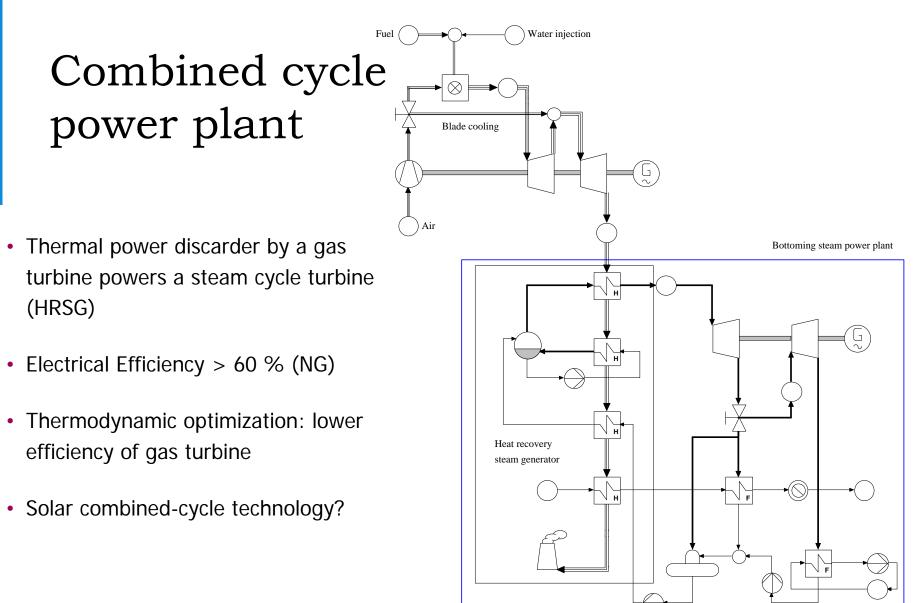




Fuel cells characteristics

- Combinations of fuel/oxydizer/electrolyte operated at difference temperatures: PEM, SOFC, MCFC,...
- The fuel cell is not a "thermal engine": the electrochemical process is isothermal
 Carnot efficiency is irrelevant
- Electrochemical reactions are more efficient than combustion + thermal engine
- Maximum theoretical efficiency of fuel cell is 100% (real is up to 60%)
- Stationary and propulsion applications but problem: initial investment cost

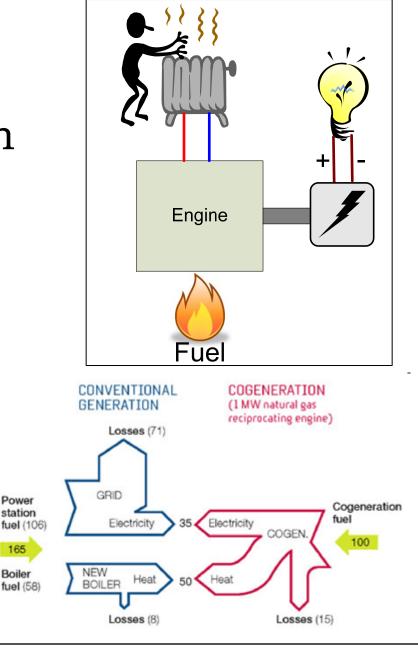






Co- and trigeneration

- Residential and process heating: temperature level
- Better use of primary energy: convert into electrical and thermal (or cooling) power at the same time
- Cogeneration of heat and power (CHP)





Example: comparison of CHP with conventional supply

INPUT DATA

Efficiencies	η
ORC turbogenerator, electrical	0.18 (18%)
ORC CHP system (el+heat)	0.96 (96%)
Average EU power plants, electrical	0.35 (35%)
Average grid losses	0.06 (6%)
Non-condensing boiler, thermal	0.87 (87%)
Power	Ŵ[ĸW]
ORC turbogenerator, electrical	120

Cogen and conventional: same electrical and thermal power output



CHP: comparison with conventional supply

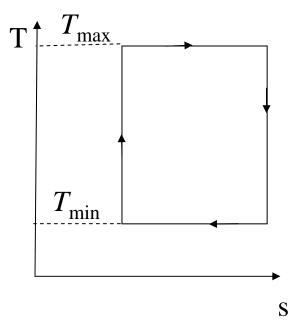
Power	Ŵ,Ż [kW]
Electrical power output	120
ORC turbogenerator, total input power	667
ORC turbogenerator, cogen thermal (90°C)	525
Input power, conventional power plant	365
Boiler, thermal power (90°C)	525
Input power, conventional boiler	603
Total input power (Power Plant + boiler)	968

Efficiency gain of CHP system = 31.1 %



Exergy Analysis: basic observations

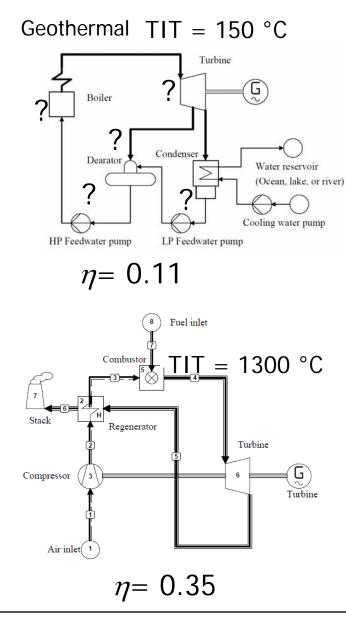
- Work if the system is not in equilibrium with the surrounding
- Maximum work between $T_{\rm max}$ and $T_{\rm min}$: Carnot cycle
- What is the maximum amount of work that can be extracted from a system (optimization)?





Tmd engineer's questions

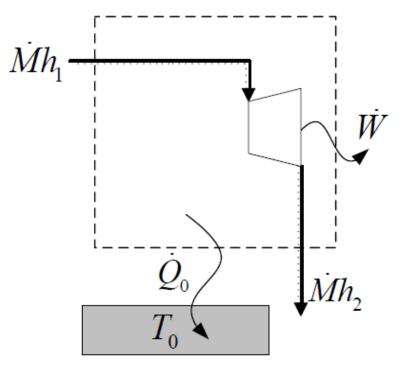
- Given a system made of components (sub-systems), which are most responsible for thermodynamic dissipation?
- Given two different energy systems operating between different temperature level, which one is operating better given its potential?





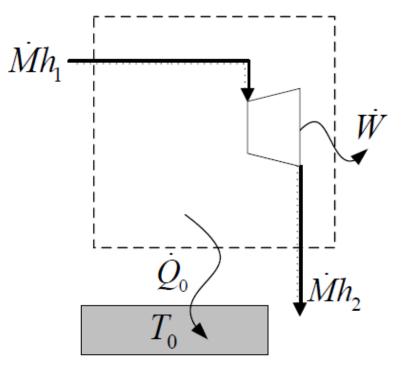
Available Energy

- Example: steady flow device
- What is the maximum power it can generate? (given some amount of energy, how much ENERGY is <u>available</u> to be converted into useful work?)
- Apply I and II law of thermodynamics



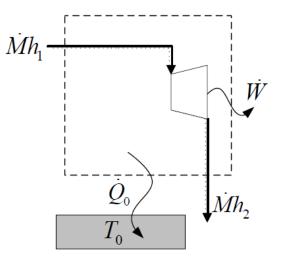






A new thermodynamic function

$$\dot{W} = \dot{M} \left[\left(h_1 - T_0 s_1 \right) - \left(h_2 - T_0 s_2 \right) \right] - T_0 \cdot \dot{\mathscr{P}}_s$$
$$e \equiv h - T_0 s$$
$$\mathcal{E} = eM$$



 \mathscr{P}_{s} : thermodynamic loss (irreversibility)

In analogy with the example: calculation of \mathscr{P}_s for every component-> primary causes of inefficiency

 \mathscr{P}_{s} 's has environmental and economic consequences

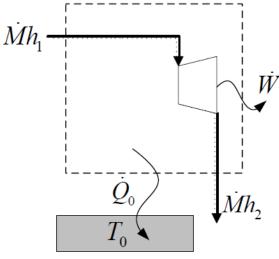


Result of the analysis

TUDelft

$$\vec{W} = \vec{M} \left[\left(h_1 - T_0 s_1 \right) - \left(h_2 - T_0 s_2 \right) \right] - T_0 \cdot \vec{\mathscr{P}}_s$$
$$e \equiv h - T_0 s$$
$$\mathcal{E} = eM$$

 \mathscr{P}_s : cannot be avoided Max \dot{W} if state 2 is at T_0 and P_0 If T_2 is greater than T_0 : cogeneration! If P_2 is greater than P_0 , add a turbine stage (P_2 must be slightly greater than P_0)



SPARE SLIDES



Thermodynamics and design (1)

- Pollution and global climate change
- Sustainable: it does not deplete permanently earth resources, (and it does not harm etc.)
- Sustainable energy systems exist: how to beat fossil-fuel based?

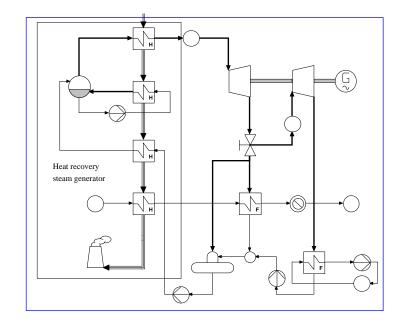


• First: reduce waste!

Thermodynamics and design (2)

Design criteria:

- Sustainability (life cycle analysis, waste, recycle measures, etc.)
- Cost (Initial, operational, etc)
- Weight
- Volume
- Aesthetics



Analysis and optimization is based on system simulations: Static and Dynamics



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The future?

- Distributed energy conversion
- Solar, wind, geothermal, biomass, tidal, OTEC, etc.
- Energy storage and batteries
- Public transportation, electric cars (batteries for baseload storage?)





